

Model Question Paper-1 with effect from 2019-20 (CBCS Scheme)

USN

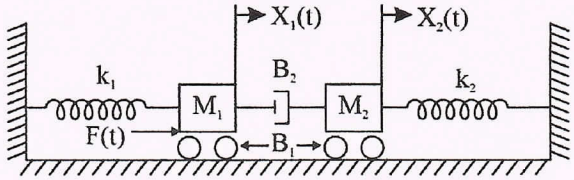
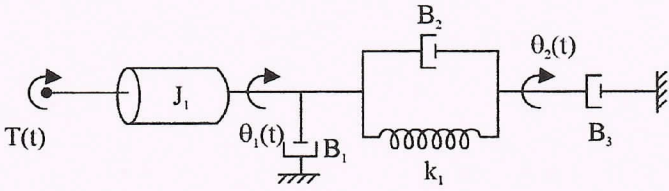
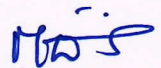
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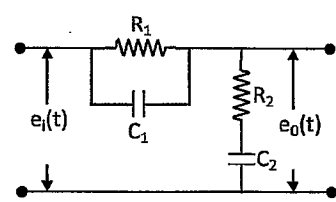
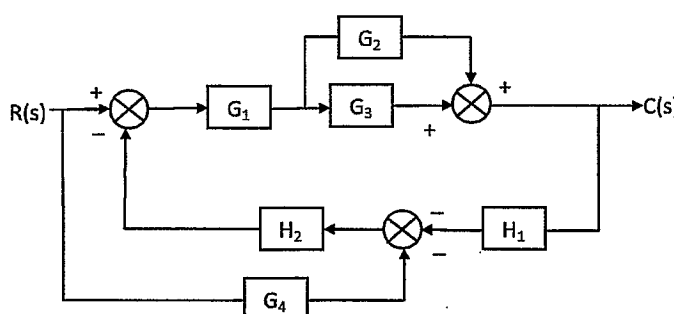
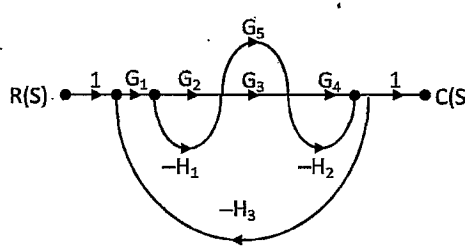
Fourth Semester B.E. Degree Examination Subject CONTROL SYSTEMS

TIME: 03 Hours

Max. Marks: 100

- Note: 01. Answer any **FIVE** full questions, choosing at least **ONE** question from each **MODULE**.
02.
03.

Module -1			*Bloom's Taxonomy Level	Marks
Q.01	a	Compare closed loop and open loop control systems. Give one example for each.	L1 CO1	06 M
	b	What are the components of a closed loop control system?	L1 CO1	04 M
	c	Find the Force Voltage analogous electrical network for the given Translational Mechanical system shown in Fig 1(c).	L1, L2 CO1	10 M
 <p style="text-align: center;">FIG 1(C)</p>				
OR				
Q.02	a	What are the effects of negative feedback in control systems?	L1 CO1	05 M
	b	What are the requirements of a good control system?	L1 CO1	05 M
	c	Find the transfer function $\theta_2(s) / T(s)$ and $\theta_1(s) / T(s)$ for the Rotational Mechanical system shown in Fig 2 c.	L1, L2 CO1	10 M
 <p style="text-align: center;">FIG 2 (C)</p>				
 Head of the Department Dept. of Electronic & Communication Engg. KLS V.D.I.T., HALIYAL (U.K.)			Submitted by Prof. Vinay Chitambar	

		Module-2			
Q. 03	a	Derive the transfer Function for the lead lag network shown in Fig 3 a. when $R_1=R_2=1\text{ M}\Omega$, $C_1 = C_2 = 1\ \mu\text{F}$.		L1 CO2	06 M
		FIG 3 (a)			
	b	Write any four rules of reducing Block diagram.		L1 CO2	04 M
	c	Find the transfer function by reducing the block diagram shown in fig 3(c).		L1, L2 CO2	10 M
		FIG 3 (c)			
OR					
Q.04	a	Define Mason's gain formula in Signal flow Graph.		L1 CO2	04 M
	b	Find Transfer function by block diagram reduction for the signal flow graph shown in Fig 4 b.		L1, L2 CO2	08 M
		FIG 4 (b)			
	c	Find the transfer function by Mason's Gain formula for the Signal flow Graph shown in Fig 4 b.		L1,L2 CO2	08 M
		Module-3			
Q. 05	a	List the standard test inputs used in control system with their Laplace transform.		L1 CO3	04 M
	b	Derive the step input response of a first order system.		L1, L2 CO3	08 M
	c	A unity negative feedback control system with $G(s) = \frac{100}{s^2(s+4)(s+12)}$ (i) What is the type of the system? (ii) Find static error coefficients. (iii) Find steady state error if the input is $r(t) = 2t^2 + 5t + 10$.		L1,L2 CO3	08 M
OR					

Q. 06	a	Starting from the output equation $C(t)$ derive expressions for: (i) Rise time (t_r) (ii) Peak overshoot (M_p) of an under damped second order system subjected to unit step input.	L1, L2 CO3	08 M
	b	For servomechanism system $G(s) = \frac{K_1}{s^2}$ and $H(s) = 1 + K_2s$. Determine the value of K_1, K_2 so that peak overshoot is 0.25 and peak time is 2 seconds for a unit step input.	L3 CO3	08 M
	c	With a neat block diagram explain PI and PD controllers.	L1 CO3	04 M
Module-4				
Q. 07	a	Explain Routh – Hurwitz criterion for stability of the system and what are its limitations.	L1 CO4	04 M
	b	Find the range of K so that system with characteristic equation as: $S^4 + 22S^3 + 10S^2 + S + K = 0$ is stable. Also find frequency of oscillation at marginal value of K.	L1,L2 CO4	06 M
	c	Plot root locus for $s^3 + 6s^2 + 8s + K = 0$	L2, CO4	10 M
OR				
Q. 08	a	For a closed loop control system, $G(s) = \frac{100}{s(s+8)}$ $H(s) = 1$, determine resonant peak and resonant frequency.	L3 CO4	04 M
	b	State any three rules of plotting root locus.	L1 CO4	06M
	c	Sketch the Bode plot for open loop transfer function. $G(s)H(s) = \frac{K}{s(1+0.2s)(1+0.05s)}$ Determine the value of K for gain margin (GM) of 6 dB.	L2,L3 CO5	10 M
Module-5				
Q. 09	a	State mapping theorem and explain Nyquist stability criterion.	L1 CO5	04 M
	b	Draw polar plot for $G(s)H(s) = \frac{10}{s(s+1)(s+2)}$ Find GM and PM, also comment on stability.	L1 CO5	10 M
	c	Explain lead lag compensating networks.	L1,CO5	06 M
OR				
Q. 10	a	What are the advantages of State Space analysis?	L1,CO5	04 M
	b	Obtain the state equations for the electrical network shown in fig 10 b.	L2, L3 CO5	08 M
<p style="text-align: center;">FIG 10 (b)</p>				
	c	Compute the STM for the system given system matrix $A = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix}$ using Laplace approach technique.	L2,L3 CO5	08 M

*Bloom's Taxonomy Level: Indicate as L1, L2, L3, L4, etc. It is also desirable to indicate the COs and POs to be attained by every bit of questions.

Model Question paper-1 (18EC43)

Module-1

Q1a. Compare closed loop and open loop control systems. Give one example for each. -06M-

Open loop	Closed loop
1) Feedback Element is absent	1) Feedback Element is present
2) Simple and Economical	2) Complex and costlier
3) Consume less Power	3) Consume more Power

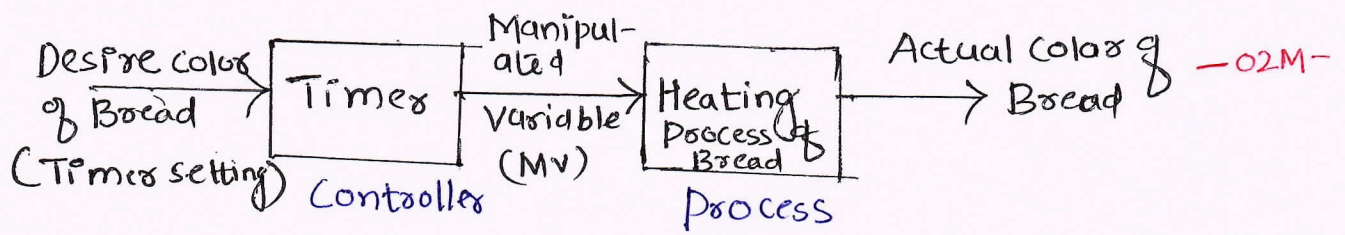
-02M-

Example of open loop control system

Bread Toaster

Objective : To toast the Bread with desire color.

Block Diagram

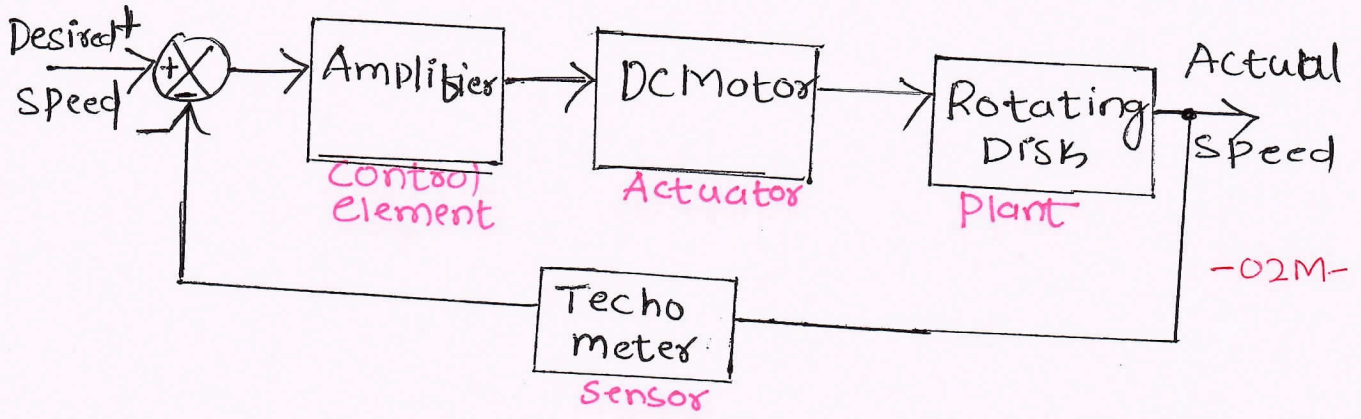


-02M-

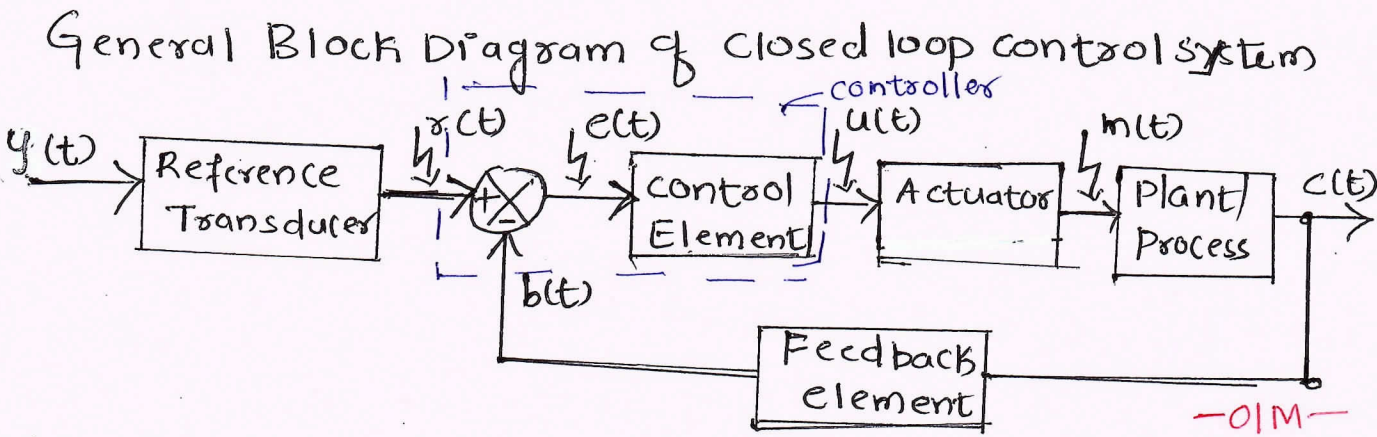
Example of closed loop control system

Rotating Disk Speed Control

Objective : To control the speed of rotating disk

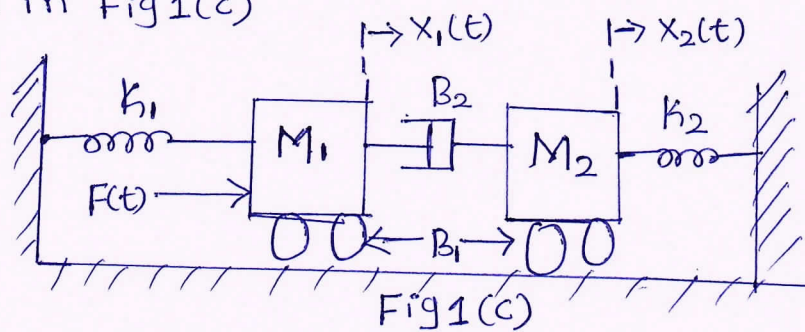


Q1b. What are the components of a closed loop control system. -04M-



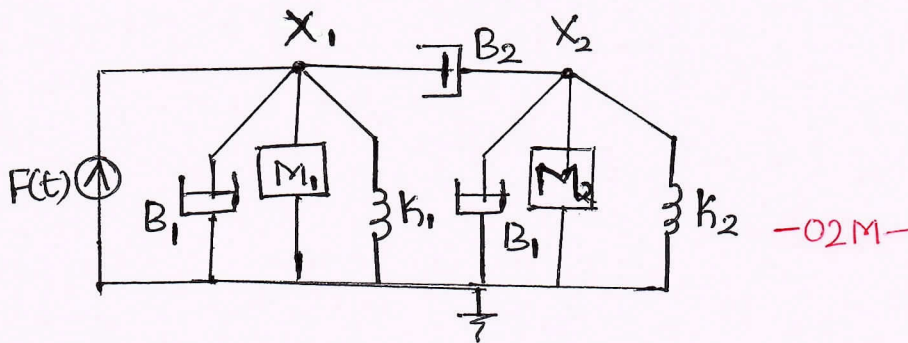
- * Plant : The system which is to be controlled or regulated is called plant / process -01M-
- * Controller : The component of control system that generates control signal to reduce the deviation of the actual value from the desired value to almost zero or lowest possible value.
- * Actuator : An actuator is device that converts energy to motion -02M
- * Feedback element : Feedback element senses the output & feed to comparator
- * Reference transducer : Converts the ^(Command) Input signal ⁱⁿ to suitable form for controller
- * $e(t)$ = error signal * $u(t)$ = control signal * $m(t)$ = manipulated variable, * $c(t)$ = controlled variable -01M-
 * $b(t)$ = feedback signal

Q1C. Find the Force Voltage analogous electric network for the given translational Mechanical system shown in Fig 1(c)



-10M-

Step 1 Nodal Diagram



-02M-

Step 2 Newton's Law at x_1

$$F(t) = M_1 \ddot{x}_1 + k_1 x_1 + B_1 \dot{x}_1 + B_2 (\dot{x}_1 - \dot{x}_2) \quad \text{--- (1)}$$

Newton's Law at x_2

$$M_2 \ddot{x}_2 + k_2 x_2 + B_1 \dot{x}_2 + B_2 (\dot{x}_2 - \dot{x}_1) = 0 \quad \text{--- (2)}$$

-03M-

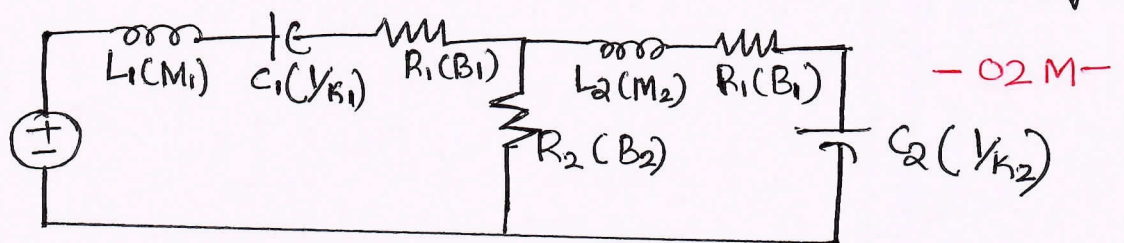
Step 3 Equivalent Electrical Ckt D.E eqn based on F-v analogy

$$V = L_1 \dot{q}_1 + \frac{1}{C_1} q_1 + R_1 \dot{q}_1 + R_2 (\dot{q}_1 - \dot{q}_2) \quad \text{--- (3)}$$

$$L_2 \dot{q}_2 + \frac{1}{C_2} q_2 + R_1 \dot{q}_2 + R_2 (\dot{q}_2 - \dot{q}_1) = 0 \quad \text{--- (4)}$$

-03M-

Step 4 Equivalent electrical circuit based on F-v analogy



-02M-

Q2a.

-OR-

What are the effects of Negative feedback -05M.

- ① The feedback reduces the effect of the disturbance with presence of feedback; the system output due to noise acting alone is

$$Y = \left(\frac{G_2}{1+G_1G_2H} \right) \underline{n} \quad \text{--- noise signal}$$

Without feedback

$$Y = G_2 n$$

-01M-

- ② The overall gain of the system reduces due to feedback.

$$Y = \left(\frac{G}{1+GH} \right) r \quad \text{--- with feedback.}$$

-01M-

- ③ The feedback reduces the sensitivity of the control systems to the parameter variation

$$S_G^M = \frac{\partial M}{\partial G} \cdot \frac{G}{M} = \frac{1}{GH+1}$$

-02M-

If GH is +ve mag of sensitivity can be made small by increasing GH .

- ④ The feedback increases the stability of system, by adding feed path with Gain ' F ' ^{for unstable system.} The overall system can be ^{made} stable by properly selecting the outer loop feedback gain ' F '

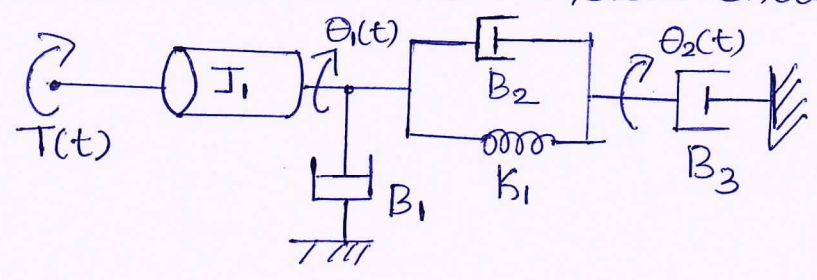
-01M-

Q2b.

What are the requirements of a good control system. -05M

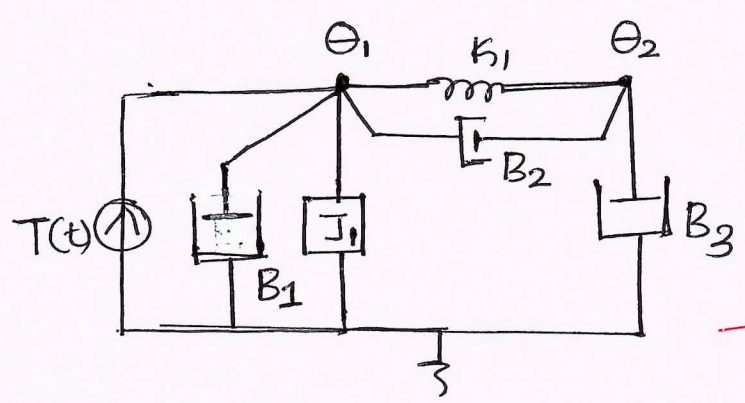
- ① Accuracy \div A Good control system must be highly accurate. -01M-
- ② Sensitivity \div A Good control system should be very insensitive to parameter variations but sensitive to the input commands. -01M-
- ③ Stability \div A Good control system should be stable over the variation of parameter in system. -01M-
- ④ Bandwidth \div A Good control system should give satisfactory output over the range of input frequency. -01M-
- ⑤ Oscillations \div A Good control system should exhibit suitable damping. -01M-

Q2c Find the transfer function $\Theta_2(s)/T(s)$ and $\Theta_1(s)/T(s)$ for the Rotational Mechanical System shown in Fig 2(c)



-10M-

Step 1 Nodal Diagram



-02M-

Step 2 Newton's Law at θ_1

$$T(t) = J_1 \ddot{\theta}_1 + B_1 \dot{\theta}_1 + K_1(\theta_1 - \theta_2) + B_2(\dot{\theta}_1 - \dot{\theta}_2) \quad \text{--- (1) ---}$$

Newton's Law at θ_2

$$K_1(\theta_2 - \theta_1) + B_2(\dot{\theta}_2 - \dot{\theta}_1) + B_3 \dot{\theta}_2 = 0 \quad \text{--- (2) ---}$$

-02M-

Step 3 let us take Laplace Transform of Eqn (1) & Eqn (2)

$$T(s) = J_1 s^2 \Theta_1(s) + B_1 s \Theta_1(s) + K_1(\Theta_1(s) - \Theta_2(s)) + B_2 s(\Theta_1(s) - \Theta_2(s)) \quad \text{--- (3) ---}$$

-02M-

$$0 = K_1(\Theta_2(s) - \Theta_1(s)) + B_2 s(\Theta_2(s) - \Theta_1(s)) + B_3 s \Theta_2(s) \quad \text{--- (4) ---}$$

Step 4 Rearranging eq (3) & solving for $\Theta_1(s)$ from eqn (4)

$$T(s) = \Theta_1(s) [J_1 s^2 + s(B_1 + B_2) + K_1] - \Theta_2(s) [B_2 s + K_1] \quad \text{--- (3a) ---}$$

$$\theta_2(s) [k_1 + s(B_2 + B_3)] - \theta_1(s) [k_1 + B_2 s] = 0$$

$$\theta_1(s) = \frac{[k_1 + s(B_2 + B_3)] \theta_2(s)}{k_1 + B_2 s} \quad \text{--- (5)}$$

By

$$\theta_2(s) = \frac{[k_1 + B_2 s] \theta_1(s)}{k_1 + s(B_2 + B_3)} \quad \text{--- (6)}$$

Steps subⁿ of eqⁿ (5) in (3a)

$$T(s) = \frac{[J_1 s^2 + s(B_1 + B_2) + k_1] [k_1 + s(B_2 + B_3)] \theta_2(s)}{k_1 + B_2 s} - \theta_2(s) [B_2 s + k_1]$$

$$T(s) = \frac{[J_1 s^2 + s(B_1 + B_2) + k_1] [k_1 + s(B_2 + B_3)] \theta_2(s)}{k_1 + B_2 s} - \theta_2(s) [B_2 s + k_1]$$

$$T(s) = \theta_2(s) \frac{[J_1 s^2 + s(B_1 + B_2) + k_1] [k_1 + s(B_2 + B_3)] - [B_2 s + k_1]^2}{k_1 + B_2 s}$$

$$\frac{\theta_2(s)}{T(s)} = \frac{k_1 + B_2 s}{(J_1 s^2 + s(B_1 + B_2) + k_1) (k_1 + s(B_2 + B_3)) - (B_2 s + k_1)^2}$$

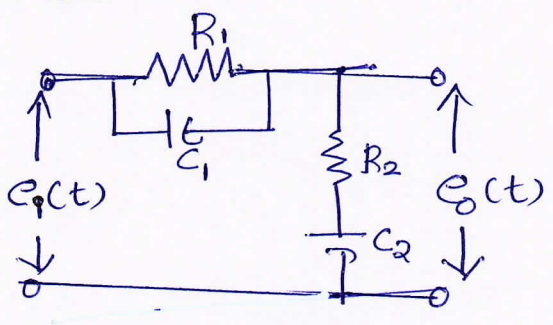
Steps subⁿ of eqⁿ (6) in (3a)

$$T(s) = (J_1 s^2 + s(B_1 + B_2) + k_1) \theta_1(s) - \frac{(k_1 + B_2 s)^2}{k_1 + s(B_2 + B_3)} \theta_1(s)$$

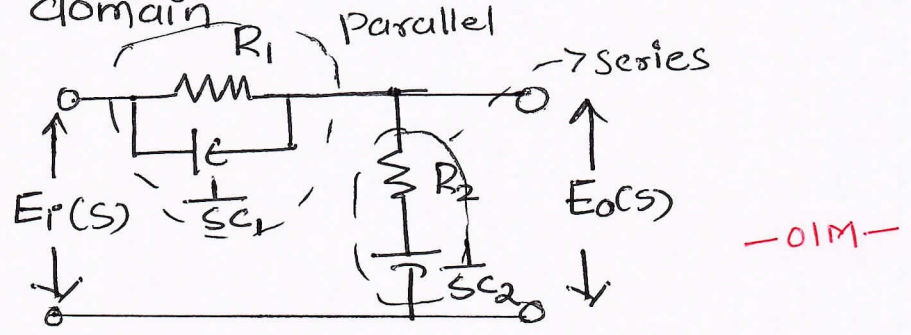
$$\frac{\theta_1(s)}{T(s)} = \frac{k_1 + s(B_2 + B_3)}{(J_1 s^2 + s(B_1 + B_2) + k_1) (k_1 + s(B_2 + B_3)) - (k_1 + B_2 s)^2}$$

Module-2

Q3a. Derive the transfer function for the lead lag network shown in Fig 3a. When $R_1 = R_2 = 1M\Omega$, $C_1 = C_2 = 1\mu F$ -06M-



Step 1 Transform the given ckt in Laplace domain

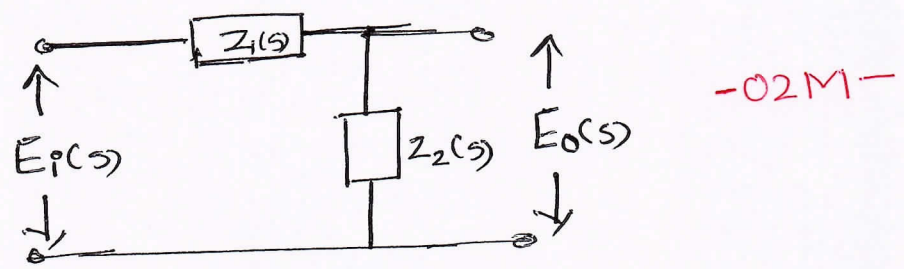


Step 2 Combining parallel and series elements

$$Z_1(s) = \frac{R_1 \times \frac{1}{sC_1}}{R_1 + \frac{1}{sC_1}} \quad \Bigg| \quad Z_1(s) = \frac{R_1}{sC_1 R_1 + 1}$$

$$Z_2(s) = R_2 + \frac{1}{sC_2} \quad \Bigg| \quad Z_2(s) = \frac{sC_2 R_2 + 1}{sC_2}$$

Step 3 Equivalent ckt after series and parallel combn



Step 4 Apply voltage divider

$$E_o(s) = E_i(s) \left[\frac{SC_2 R_2 + 1}{SC_2} \right] \quad \Bigg| \quad E_o(s) = \frac{E_i(s) Z_2(s)}{Z_1(s) + Z_2(s)}$$

$$\frac{R_1}{SC_1 R_1 + 1} + \frac{SC_2 R_2 + 1}{SC_2} \quad \Bigg|$$

$$E_o(s) = \frac{E_i(s) (SC_2 R_2 + 1)}{\frac{SC_2 R_1}{SC_1 R_1 + 1} + SC_2 R_2 + 1} = \frac{E_i(s) (SC_2 R_2 + 1) (SC_1 R_1 + 1)}{SC_2 R_1 + (SC_2 R_2 + 1) (SC_1 R_1 + 1)}$$

-02M-

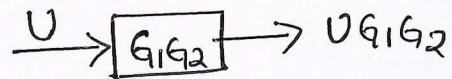
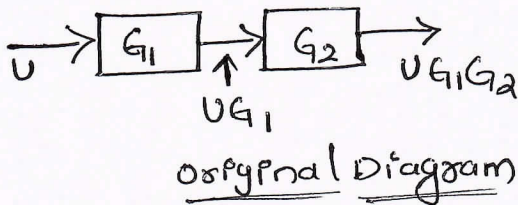
$$C_1 R_1 = 1, \quad C_2 R_2 = 1, \quad R_1 C_2 = 1$$

$$\frac{E_o(s)}{E_i(s)} = \frac{(s+1)^2}{(s+1)^2 + s}$$

-01M-

Q3b. Write any four rules of Reducing Block diagram -04M-

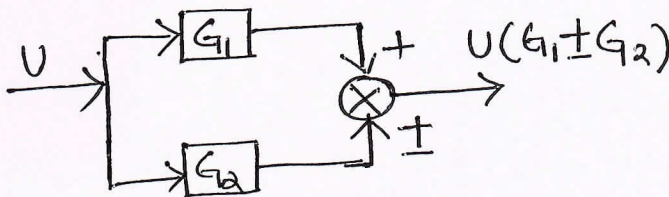
① Combining Blocks in cascade



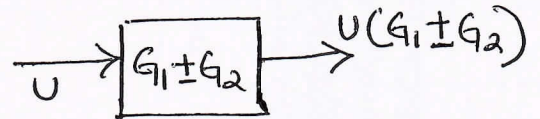
-01M-

② Combining Blocks in parallel

Original diagram



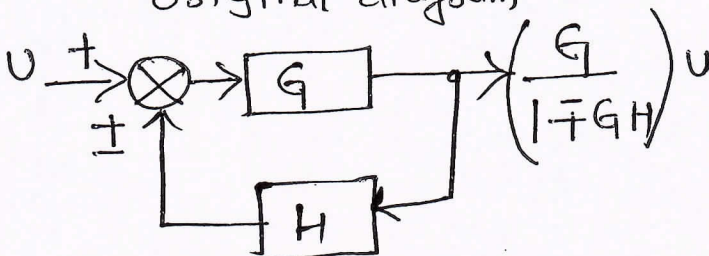
Equivalent diagram



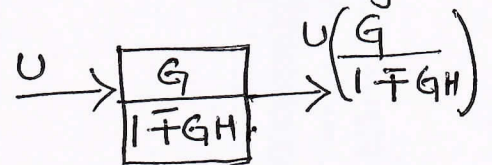
-01M-

③ Eliminating feedback loop

Original diagram



Equivalent diagram

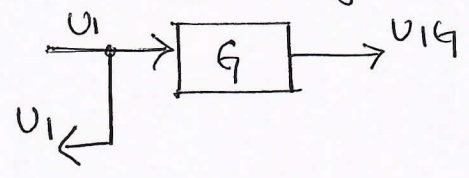


-01M-

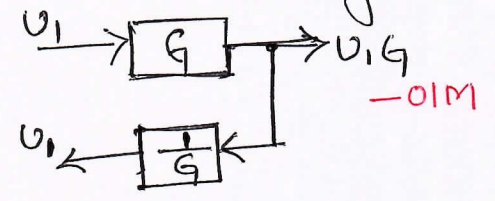
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Moving Take off Point after Block

Original diagram

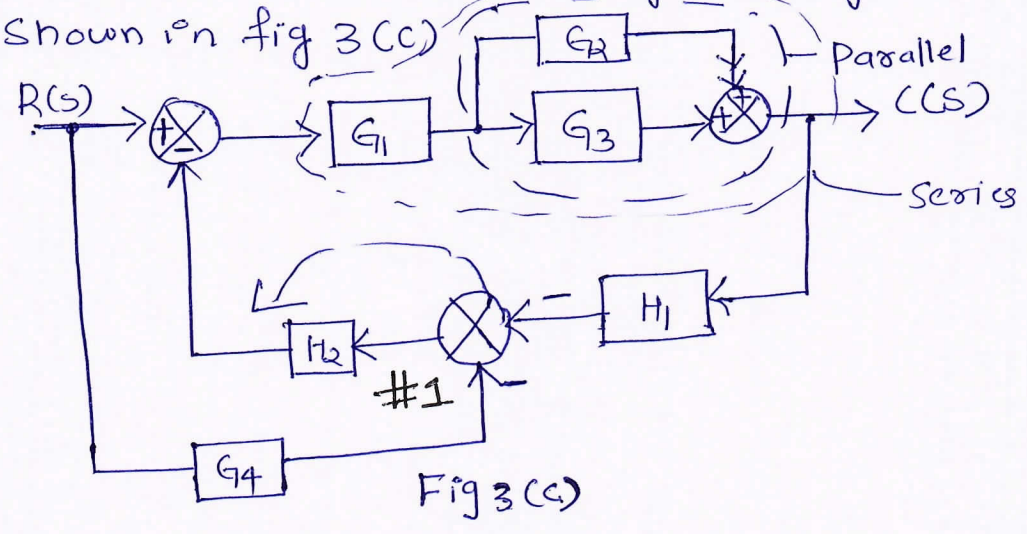


Equivalent diagram



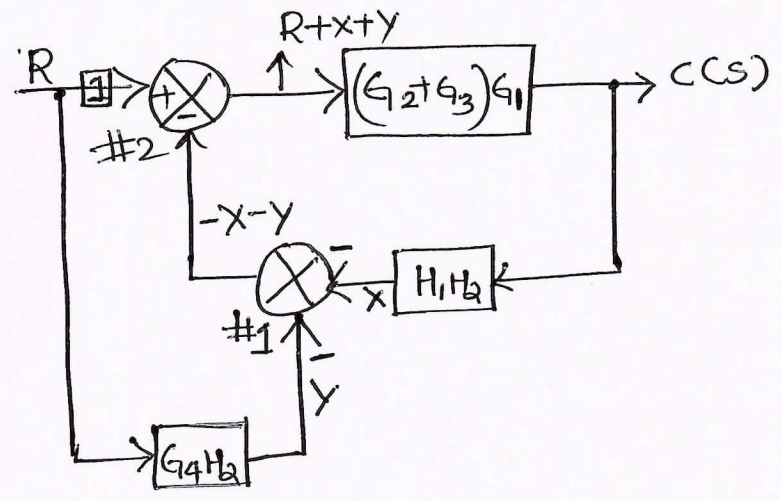
Q3C.

Find the transfer function by reducing the block diagram shown in fig 3(c)



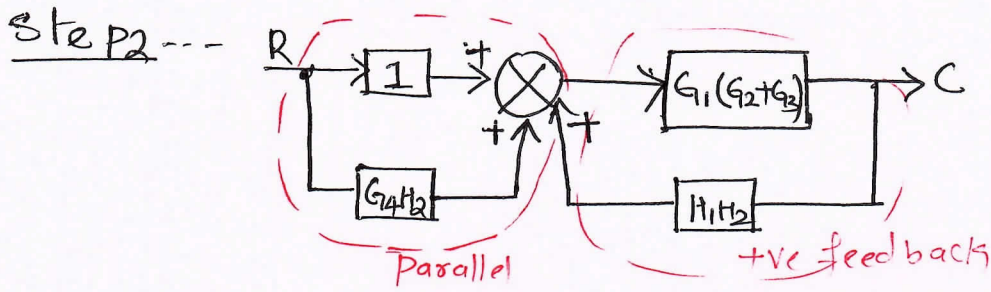
Step 1

combining G_2 & G_3 block in parallel ($G_2 + G_3$), parallel combination is combined in series with G_1 ($(G_2 + G_3)G_1$), shifting summer #1 in forward direction.

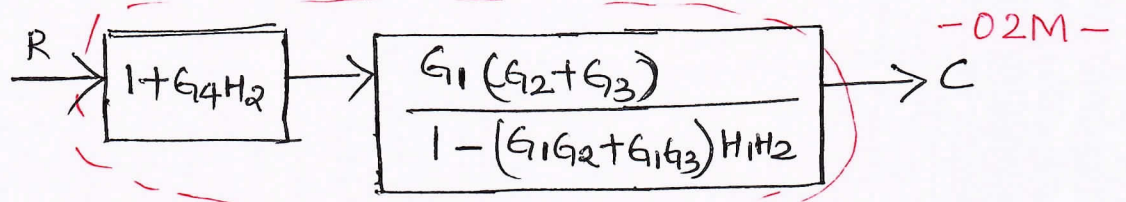


Step 2

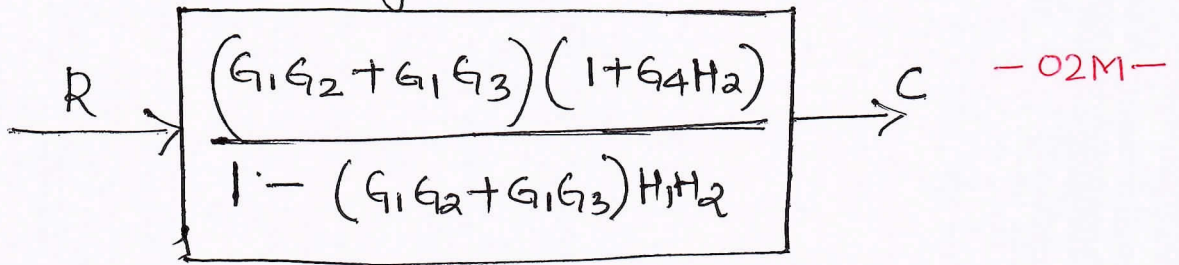
Combining summer #1 and #2



Step 3 Combining parallel block G_4H_2 with gain of 1 & eliminating +ve feedback



Step 4 Combining Blocks in Series



-OR-

Q4a. Define Mason's gain formula in signal flow graph. -04M-

Mason's gain formula for the determination of the overall system gain (Transfer function) is given by

$$T = \frac{1}{\Delta} \sum_{k=1}^N P_k \Delta_k \quad | \quad T = \text{Overall gain of the system (Transfer function)}$$

-02M-

Δ : Determinant of the graph

$$\Delta = 1 - \left(\text{sum of individual loop gain} \right) + \left(\text{sum of gain products of all comb}^n \text{ of two non-touching loops} \right) - \left(\text{sum of gain products of all comb}^n \text{ of three non-touching loops} \right) + \dots$$

N : No. of forward path in the signal flow graph

-02M-

Δ_k : The value of delta for the part of the graph not touching the k^{th} forward path

P_k : Path gain of k^{th} forward path

Q4b. Find Transfer function by Block diagram reduction for the signal flow graph shown in Fig 4(b)

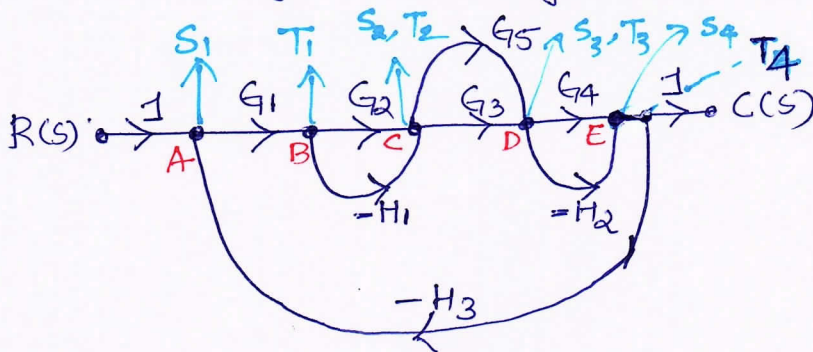
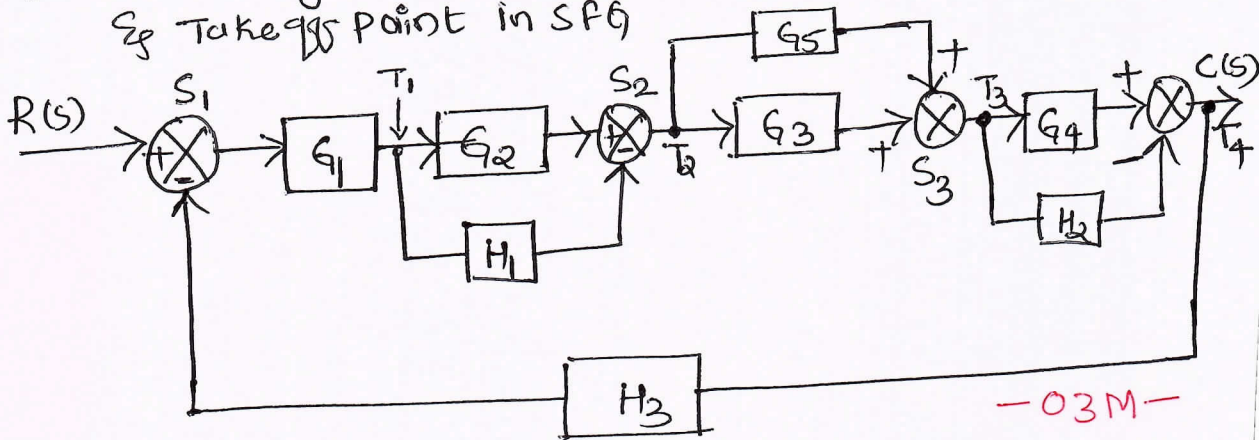


Fig 4(b)

-08M-

Step 1 : Identifying Summers & Takeoff points in SFG

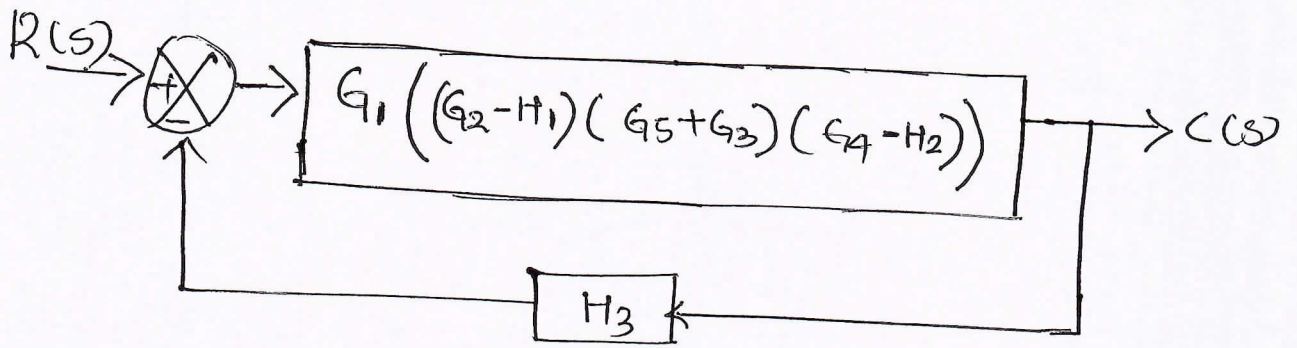
Step 2 : Drawing Block diagram with identical summer & Takeoff point in SFG



-03M-

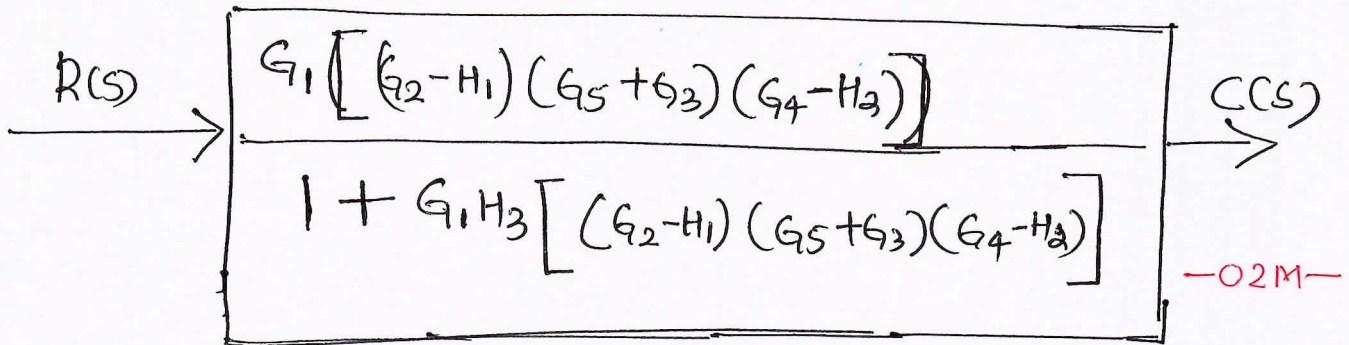
Step 3 : Reducing Block diagram, (G_2, H_1) , (G_5, G_3) , (G_4, H_2) are all in parallel $(G_2 - H_1)$, $(G_5 + G_3)$, $(G_4 - H_2)$, this whole parallel combⁿ is series with each other. This whole combⁿ is series with G_1 .

-03M-



-02M-

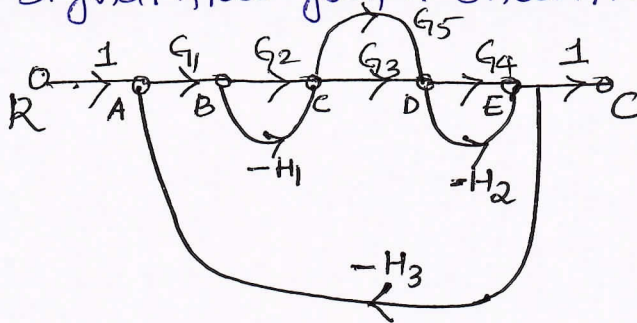
Step 4 Eliminating -ve feedback



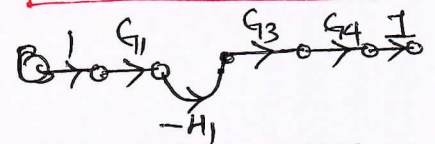
-02M-

Q4C Find the Transfer function by Mason's Gain formula for the signal flow graph shown in Fig (b)

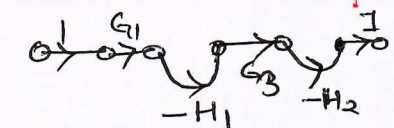
-08M-



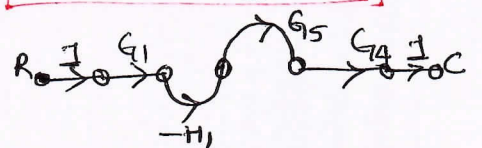
$$P_4 = -G_1 G_2 G_5 H_2$$



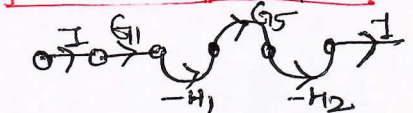
$$P_5 = -G_1 G_3 G_4 H_1$$



$$P_6 = G_1 G_3 H_1 H_2$$

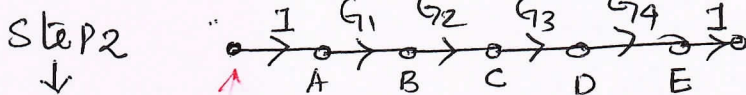


$$P_7 = -G_1 G_5 G_4 H_1$$



$$P_8 = G_1 G_5 H_1 H_2$$

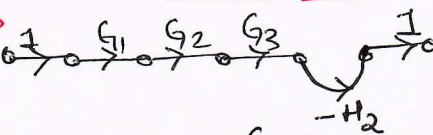
Step 1 No' of forward path $N=8$



[Individual forward path with its gain]

$$P_1 = G_1 G_2 G_3 G_4$$

$$P_2 = -G_1 G_2 G_3 H_2$$

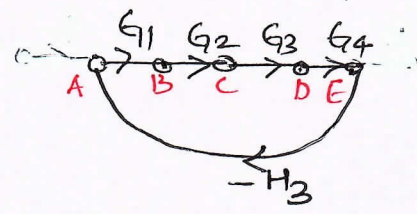


$$P_3 = G_1 G_2 G_5 G_4$$

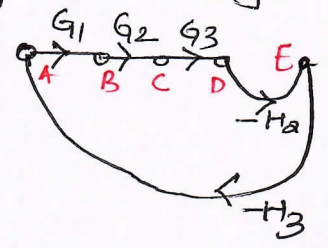
-03M-

STEP 3

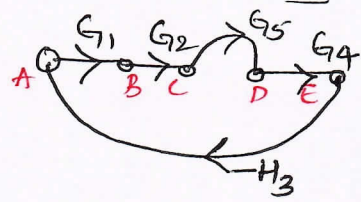
Individual loops along with its gain



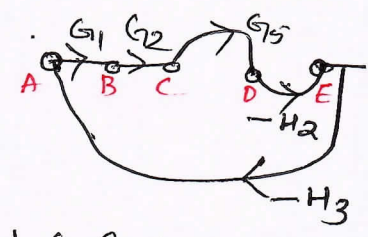
$$L_1 = -G_1 G_2 G_3 G_4 H_3$$



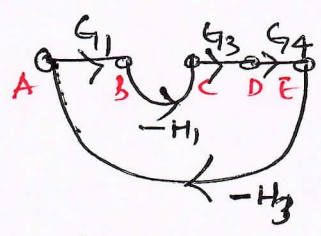
$$L_2 = +G_1 G_2 G_3 H_2 H_3$$



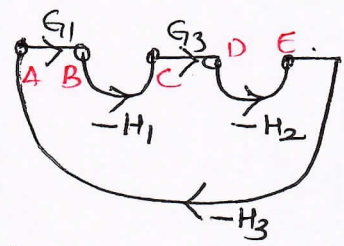
$$L_3 = -G_1 G_2 G_4 G_5 H_3$$



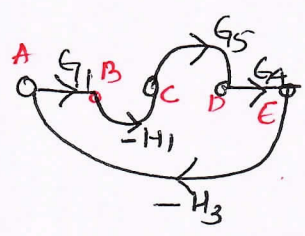
$$L_4 = G_1 G_2 G_5 H_2 H_3$$



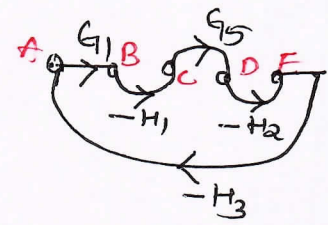
$$L_5 = G_1 G_3 G_4 H_1 H_3$$



$$L_6 = -G_1 G_3 H_1 H_2 H_3$$



$$L_7 = G_1 G_4 G_5 H_1 H_3$$



$$L_8 = -G_1 G_5 H_1 H_2 H_3$$

-O3M-

Step 4

Non touching loops $\hat{=}$ All loop have common nodes A, B, C, D, E. \therefore There is no combⁿ of Non-touching loop - NIL

Steps : $\Delta = 1 - [L_1 + L_2 + L_3 + L_4 + L_5 + L_6 + L_7 + L_8]$

Step 6 : Δ_k : All forward path and all loops have common node $\therefore \Delta_1 = \Delta_2 = \Delta_3 = \Delta_4 = \Delta_5 = \Delta_6 = \Delta_7 = \Delta_8 = 1$

-O1M-

Step 7 $T_o F = \frac{1}{\Delta} \sum_{k=1}^8 P_k \Delta_k \quad | \quad \Delta_k = 1 \quad | \quad k \text{ varies from } 1-8$

$$= \frac{P_1 + P_2 + P_3 + P_4 + P_5 + P_6 + P_7 + P_8}{\Delta}$$

$$T_o F = \frac{G_1 G_2 G_3 G_4 - G_1 G_2 G_3 H_2 + G_1 G_2 G_5 G_4 - G_1 G_2 G_5 H_2 - G_1 G_3 G_4 H_1 + G_1 G_3 H_1 H_2 - G_1 G_5 G_4 H_1 + G_1 G_5 H_1 H_2}{\Delta} \quad \text{--- 01M ---}$$

$$\Delta = 1 + G_1 G_2 G_3 G_4 H_3 - G_1 G_2 G_3 H_2 H_3 + G_1 G_2 G_4 G_5 H_3 - G_1 G_2 G_5 H_2 H_3 - G_1 G_3 G_4 H_1 H_3 + G_1 G_3 H_1 H_2 H_3 - G_1 G_4 G_5 H_1 H_3 + G_1 G_5 H_1 H_2 H_3$$

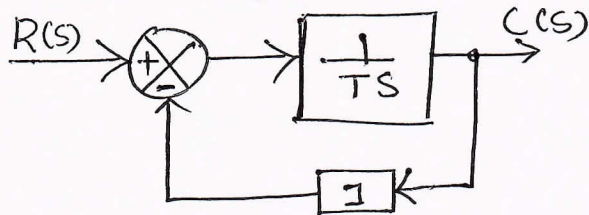
--- Module-3 ---

Q5a. List the standard test inputs used in control system with their Laplace transform. --- 04M ---

Test signals	Laplace transform of test signal	
1) Unit Impulse $\delta(t)$	1	--- 01M ---
2) Unit step $u(t)$	$\frac{1}{s}$	--- 01M ---
3) Unit Ramp $x(t) = t$	$\frac{1}{s^2}$	--- 01M ---
4) Unit Parabolic $x(t) = \frac{t^2}{2}$	$\frac{1}{s^3}$	--- 01M ---

Q5b. Derive the step input response of a First order System. -08M-

Standard Block Diagram of a first order system is given by



* where 'T' is called time constant of system

-01M-

Step 1 Transfer function of above Block Diagram

$$\frac{C(s)}{R(s)} = \frac{1}{Ts+1}$$

Step 2 Solving for o/p from T.F

$$C(s) = \frac{1}{Ts+1} \cdot R(s)$$

(o/p) (T.F) (i/p)

Step 3 For unit step resp $R(s) = \frac{1}{s}$, subⁿ of $R(s)$ in above o/p eqⁿ

$$C(s) = \frac{1}{Ts+1} \cdot \frac{1}{s} \quad \text{-03M-}$$

Step 4 Apply partial fraction expansion for above o/p eqⁿ

$$C(s) = \frac{k_1}{Ts+1} + \frac{k_2}{s} \quad \left| \quad k_1 = \frac{1}{s} \right|_{s=-\frac{1}{T}}$$

$$k_2 = \frac{1}{Ts+1} \Big|_{s=0} = 1 \quad \left| \quad \boxed{k_1 = -T} \right. \quad \left. \boxed{k_2 = 1} \right. \quad \text{-03M-}$$

$$C(s) = \frac{-T}{Ts+1} + \frac{1}{s}$$

$$\therefore C(s) = \frac{1}{s} - \frac{1}{s+\frac{1}{T}}$$

Step 5 Inverse Laplace of above o/p eqⁿ

$$C(t) = (1 - e^{-t/T}) u(t)$$

-01M-

Q5C.

A unity negative feedback control system with

$$G(s) = \frac{100}{s^2(s+4)(s+12)}$$

(i) What is the type of system

(ii) Find static error coefficient

(iii) Find steady state error if the r/p is $x(t) = 2t^2 + 5t + 10$

-08M-

Step 1 (i) Type of system is no. of poles at origin from above ^{Given} $G(s)$, the no. of poles at origin is '2' \therefore Type of system is 2

-01M-

Step 2 (ii) Static error coefficients: 'K_p' position error

$$\text{Constant } K_p = \lim_{s \rightarrow 0} G(s) = \lim_{s \rightarrow 0} \frac{100}{s^2(s+4)(s+12)}$$

$$\boxed{K_p = \infty}$$

'K_v' velocity error constant

$$K_v = \lim_{s \rightarrow 0} sG(s) = \lim_{s \rightarrow 0} \frac{s \cdot 100}{s^2(s+4)(s+12)} = 0$$

$$\boxed{K_v = \infty}$$

'K_a' acceleration error constant

$$K_a = \lim_{s \rightarrow 0} s^2 G(s) = \lim_{s \rightarrow 0} \frac{s^2 \cdot 100}{s^2(s+4)(s+12)} = \frac{100}{4 \times 12}$$

$$\boxed{K_a = \frac{25}{12}}$$

-03M-

Step 3 (iii) error for sum of three input is sum of individual error $e_{ss} = e_{ss1} + e_{ss2} + e_{ss3}$ | $A_1 = 10$ | $A_3 = 4$

$$e_{ss} = \frac{A_1}{1+K_p} + \frac{A_2}{K_v} + \frac{A_3}{K_a} \quad \left| \begin{array}{l} A_2 = 5 \\ \end{array} \right.$$

$$e_{ss} = \frac{10}{1+\infty} + \frac{5}{\infty} + \frac{4}{\frac{25}{12}}$$

$$e_{ss} = \frac{48}{25} = 1.92$$

$$\boxed{e_{ss} = 1.92}$$

-04M-

Q6a.

Starting from the output equation $c(t)$ derive expression for (i) Rise time (t_r) (ii) Peak overshoot (M_p) of under damped second order system subjected to unit step inputs -08M-

(i)

Step 1 The output expression for under damped second order system is given by

$$C(t) = 1 - \frac{e^{-\zeta\omega_n t}}{\sqrt{1-\zeta^2}} \sin(\omega_d t + \theta) \quad \text{-01M-}$$

According to defⁿ of rise time @ $t = t_r$ $c(t) = 1$

$$\therefore C(t_r) = 1 = 1 - \frac{e^{-\zeta\omega_n t_r}}{\sqrt{1-\zeta^2}} \sin(\omega_d t_r + \theta)$$

$$\sin(\omega_d t_r + \theta) = 0$$

$$\omega_d t_r + \theta = \sin^{-1}(0) = n\pi ; n = 0, 1, 2 \dots \text{ for first}$$

instant $n=1$

$$\omega_d t_r + \theta = \pi$$

Step 2 Solving for t_r from above eqⁿ

$$t_r = \frac{\pi - \theta}{\omega_d} = \frac{\pi - \theta}{\omega_n \sqrt{1-\zeta^2}} \quad \left. \begin{array}{l} \text{where } \theta = \cos^{-1} \zeta \\ \text{-02M-} \end{array} \right\}$$

(ii) According to defⁿ of Peak overshoot

$$M_p = \frac{C(t_p) - C(\infty)}{C(\infty)}, \text{ for unit step i/p } C(\infty) = 1 \quad \text{-01M-}$$

$$\therefore M_p = C(t_p) - 1$$

$$\therefore C(t_p) = 1 - \frac{e^{-\zeta\omega_n t_p}}{\sqrt{1-\zeta^2}} \sin(\omega_d t_p + \theta)$$

$$M_p = C(t_p) - 1 = -\frac{e^{-\zeta\omega_n t_p}}{\sqrt{1-\zeta^2}} \sin(\omega_d t_p + \theta)$$

Subⁿ of t_p expression $t_p = \frac{\pi}{\omega_d}$ in M_p

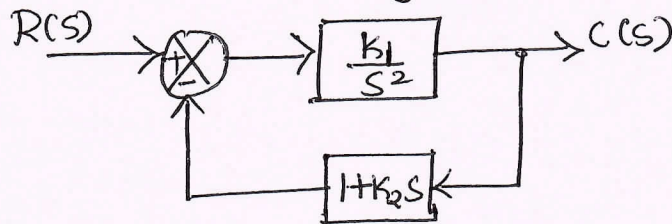
$$M_p = \frac{-e^{-\zeta \omega_n \frac{\pi}{\omega_d}} \cdot \sin\left(\omega_d \cdot \frac{\pi}{\omega_d} + \theta\right)}{\sqrt{1-\zeta^2}} \quad -02M-$$

$$= \frac{-e^{-\zeta \omega_n \frac{\pi}{\omega_d \sqrt{1-\zeta^2}}} \cdot (-\sin\theta)}{\sqrt{1-\zeta^2}}$$

$$M_p = \frac{e^{-\zeta \pi / \sqrt{1-\zeta^2}} \cdot \sin\theta}{\sqrt{1-\zeta^2}} \quad \left| \quad \begin{array}{l} \text{but } \sin\theta = \sqrt{1-\zeta^2} \\ \hline M_p = e^{-\zeta \pi / \sqrt{1-\zeta^2}} \end{array} \right. \quad -02M-$$

Q6b. For a servomechanism system with $G(s) = \frac{k_1}{s^2}$ and $H(s) = 1+k_2s$. Determine the value of k_1, k_2 so that peak overshoot is 0.25 and peak time is 2 seconds for a unit step input. -08M-

Step 1 The Block Diagram for given system is



$$\frac{C(s)}{R(s)} = \frac{k_1}{s^2 + k_1(1+k_2s)} = \frac{k_1}{s^2 + k_1k_2s + k_1} \quad -02M-$$

Step 2 standard 2nd order T.F = $\frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}$

Comparing standard T.F with obtained T.F

$$\omega_n^2 = k_1 \quad 2\zeta\omega_n = k_1k_2 \quad -②-$$

Step 3 Given $M_p = 0.25$ (25%) we can solve for

$$\zeta, \quad \zeta = \frac{-\ln(0.25)}{\sqrt{\pi^2 + \ln^2(0.25)}}$$

$$\boxed{\zeta = 0.403}$$

$$T_p = \frac{\pi}{\omega_n \sqrt{1-\zeta^2}} = 2$$

$$\omega_n = \frac{\pi}{2\sqrt{1-\zeta^2}}$$

$$\boxed{\omega_n = 1.716 \text{ rad/s}} \quad -0.4M-$$

Step 4 Subⁿ of ζ & ω_n in eqⁿ ① & ② & solving for k_1 and k_2

$$k_1 = \omega_n^2 = (1.716)^2 = 2.947$$

$$\boxed{k_1 = 2.947}$$

$$2\zeta\omega_n = k_1 k_2$$

$$k_2 = \frac{2 \times 0.403 \times 1.716}{2.947}$$

$$\boxed{k_2 = 0.469}$$

$$\therefore \boxed{k_1 = 2.947, k_2 = 0.469} \quad -0.1M-$$

Q6.c With neat Block diagram explain PI and PD Controller -0.4M-

Proportional-Integral Controller (PI Controller)

The controller with PI control action, the relationship between the output of controller $u(t)$ and the actuating error signal $e(t)$ is given by

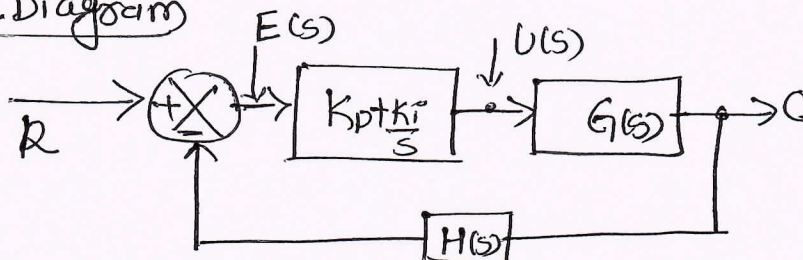
$$u(t) = K_p e(t) + K_i \int_0^t e(t) dt \quad \left| \quad \frac{U(s)}{E(s)} = K_p + \frac{K_i}{s} \right.$$

Laplace Transform of above eqⁿ

$$U(s) = K_p E(s) + \frac{K_i}{s} E(s)$$

PI Controller Block is represented by Block with gain $K_p + \frac{K_i}{s}$

Block Diagram



-0.2M-

Proportional-derivative controller (PD controller)

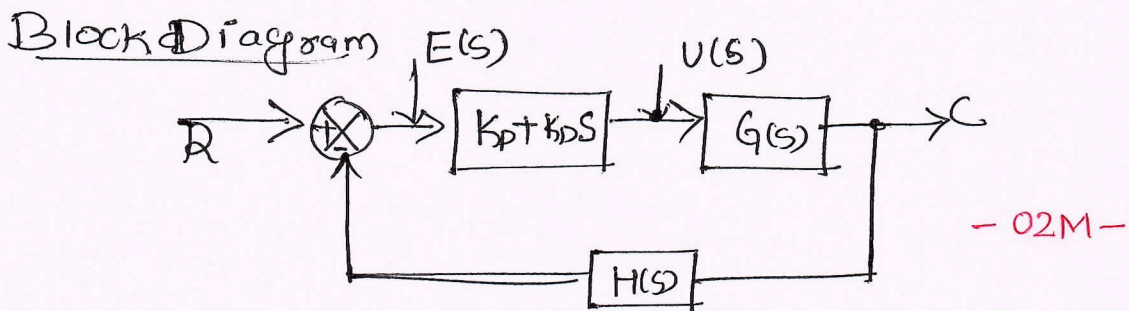
The controller with PD control action the relation between the output of controller $u(t)$ and the actuating error signal $e(t)$ is given by

$$u(t) = K_p e(t) + K_D \frac{de(t)}{dt} \quad | \quad \frac{U(s)}{E(s)} = K_p + K_D s$$

Leplace Transform of above eqn)

$$U(s) = K_p E(s) + K_D s E(s) \quad |$$

PD controller Block is represented by Block with gain $K_p + K_D s$



- Module - 4 -

Q7a Explain Routh-Hurwitz criterion for stability of the systems and what are its limitations. - 04M -

The Routh-Hurwitz criterion is necessary & sufficient condition for the stability of LTI systems

* The method requires two steps

(i) Generating Routh array

(ii) Interpreting Routh array for location of poles in the s-plane - 02M -

* The Routh-Hurwitz criterion states that the number of roots of $C E$ eqn with positive real parts is equal to the number of changes in sign of the first column of the Routh array - 01M -

* Limitations

- 1) Applicable only to linear system
- 2) It doesnot provide exact location (Co-ordinates) in LHP or RHP
- 3) It is valid ^{only} for real coefficients of the C.E equation.

-01M-

Q7b Find the range of K so that system with C.E as:
 $s^4 + 22s^3 + 10s^2 + s + K = 0$ is stable. Also find frequency of oscillation at marginal value of K.

-06M-

Step 1 ∴ Constructing (Generating) Routh table

s^4	1	10	K		$\frac{219 - 22K}{22} = \frac{219 - 484K}{219}$ $\frac{219}{22}$
s^3	22	1	0		
s^2	$\frac{219}{22}$	K	0		
s^1	$\frac{219 - 484K}{219}$	0	0		
s^0	K	0	0		

-02M-

Step 2: For system to stable all the elements of first column should be positive

$$K > 0 \quad \& \quad \frac{219 - 484K}{219} > 0 \quad \left| \quad K < \frac{219}{484} \right.$$

∴ range of K for stable

$K < 0.45$

System is $0 < K < 0.45$

Step 3 ∴ For marginal stable system, Routh table require Row of zero of A(s) ∴ from s^1 row

$$\frac{219 - 484K}{219} = 0 \Rightarrow K = \frac{219}{484} = 0.45$$

$K = 0.45 = K_{\text{marg}}$

-02M-

Step 4 $A(s) = \frac{219}{22} s^2 + k = 0 \quad | \quad k = 0.45$

$A(s) = \frac{219}{22} s^2 + 0.45 = 0$

$\therefore s^2 = \frac{-0.45 \times 22}{219}$

$s^2 = -0.045$

$s = \pm j 0.213$

$s = \pm j \omega$

$\therefore \omega = 0.213 \text{ rad/s}$

(frequency of oscillation)

$k_{max} = 0.45$

(Marginal value of k)

- 02M -

Q7C Plot root locus for $s^3 + 6s^2 + 8s + k = 0$

- 10M -

Step 1: From give C.E we need find open loop transfer function $G(s)H(s)$.

Step 2: Rewriting C.E as

$(s^3 + 6s^2 + 8s) + k = 0$

dividing above eqⁿ by $s^3 + 6s^2 + 8s$

$1 + \frac{k}{s^3 + 6s^2 + 8s} = 0$

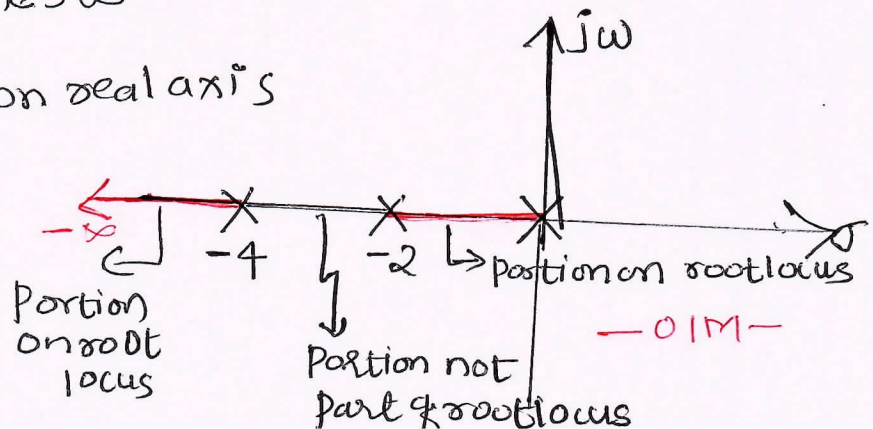
$1 + G(s)H(s) = 0$

$\therefore G(s)H(s) = \frac{k}{s^3 + 6s^2 + 8s} = \frac{k}{s(s^2 + 6s + 8)} = \frac{k}{s(s+4)(s+2)}$

- 01M -

Step 3 $P = 3, Z = 0, N = P = 3, P - Z = 3$ branches approaches ∞

Step 4 section on real axis



Step 5 Angle of asymptotes

$$\theta = \frac{(2q+1)180^\circ}{P-Z}, \quad q=0,1,2$$

$$= \frac{(2q+1)180^\circ}{3} \quad \left| \quad \theta = 60^\circ, 180^\circ, 300^\circ \right|$$

Step 6 Centroid = $\frac{\sum \text{Real part of open loop Poles} - \sum \text{Real part of open loop Zeros}}{P-Z}$

$$\sigma = \frac{0-4-2-0}{3} = -\frac{6}{3} = -2$$

$$\sigma = -2$$

-01M-

Step 7 Breakaway Point

$$1+G(s)H(s) = 1 + \frac{K}{s^3+6s^2+8s} = 0$$

$$s^3+6s^2+8s+K=0 \quad \left| \quad \frac{dK}{ds} = -3s^2-12s-8=0 \right|$$

$$K = -s^3-6s^2-8s \quad \left| \quad \therefore 3s^2+12s+8=0 \right|$$

$$s_1 = -0.845, \quad s_2 = -3.154$$

$s_2 = -3.154$ is not part of root locus on real axis, hence s_2 is not valid Breakaway point

$s_1 \Rightarrow$ is part of root locus on real axis. $\therefore s_1$ is valid Breakaway point

-02M-

Step 8 Intersection with imaginary axis

C.E $s^3+6s^2+8s+K=0$

R.H Table

s^3	1	8	
s^2	6	K	
s^1	$\frac{48-K}{6}$	0	
s^0	K		

$$\frac{48-K}{6} = 0 \quad \left| \quad K = 48 = K_{max} \right|$$

-02M-

$$A(s) = 6s^2 + K = 0$$

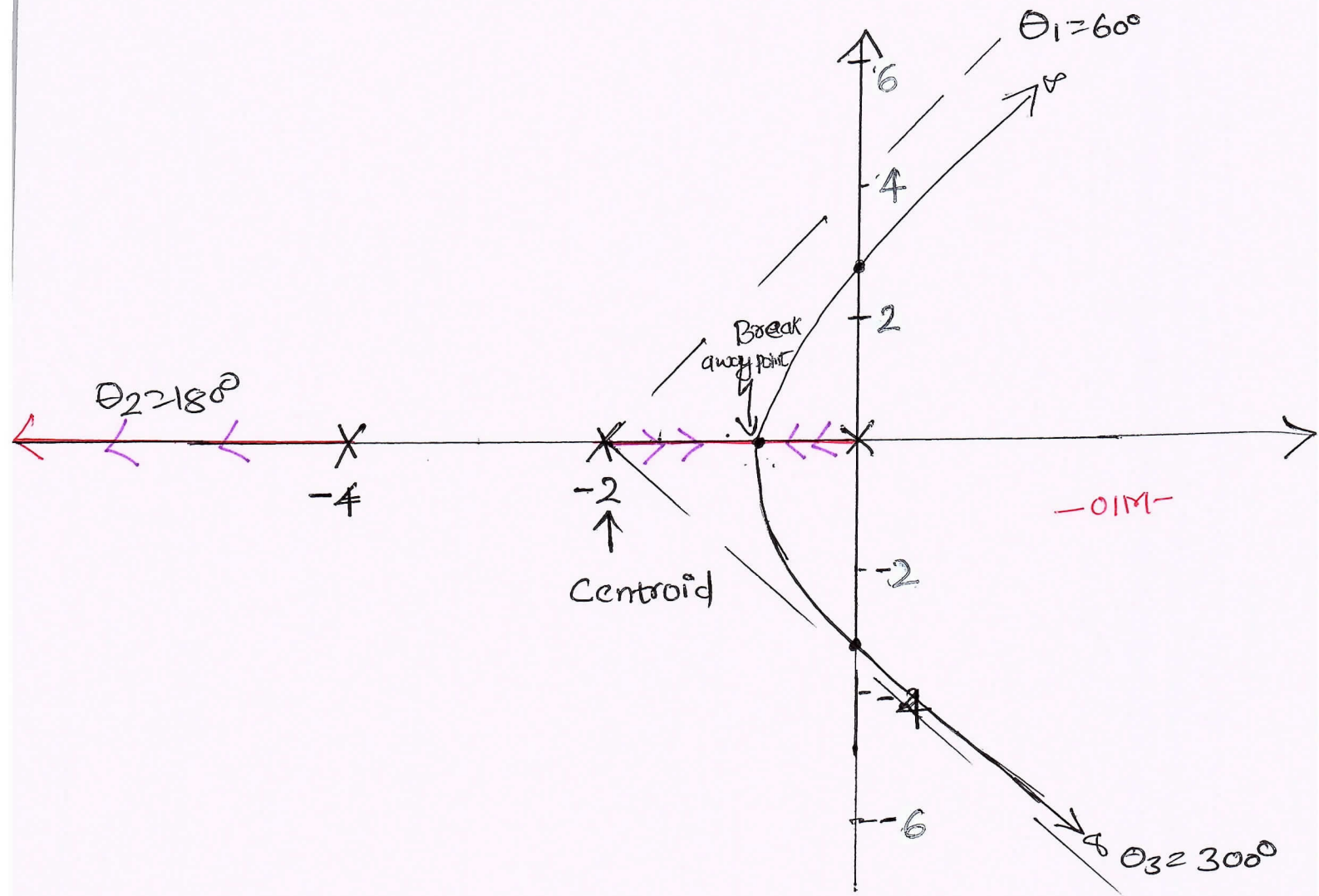
$$6s^2 + 48 = 0$$

$$s^2 = -8 \quad \left| \quad s = \pm j\sqrt{8} \right|$$

$$\omega = 2.828 \text{ rad/s}$$

-02M-

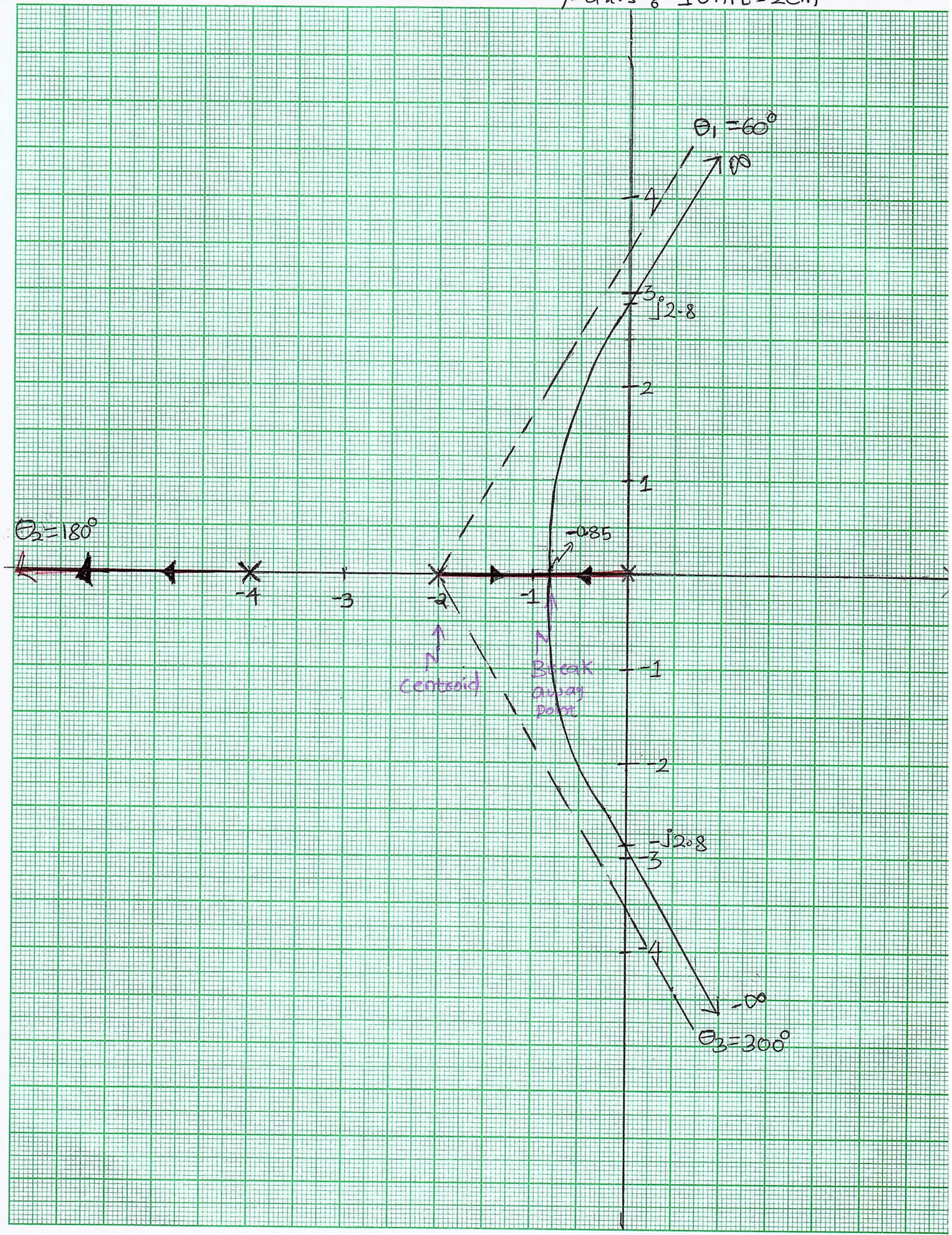
* The complete Root locus ^{can be} plotted with above (25)
Obtained information



QTC -02M-

Scale: X-axis: 1 unit = 2cm
Y-axis: 1 unit = 2cm

(26)



Q8a. For a closed loop control system, $G(s) = \frac{100}{s(s+8)}$, $H(s) = 1$, determine resonant peak and resonant frequency. -04M-

Step 1 $\hat{=}$ Finding Transfer function

$$\frac{C(s)}{R(s)} = \frac{G(s)}{1+G(s)} = \frac{100}{s^2+8s+100}$$

Step 2: Comparing with standard 2nd order T.F

$$\frac{\omega_n^2}{s^2+2\zeta\omega_n s+\omega_n^2} \quad \left| \quad \begin{array}{l} \omega_n^2=100 \\ 2\zeta\omega_n=8 \end{array} \right. \quad \left| \quad \begin{array}{l} \omega_n=10 \\ \zeta = \frac{8}{2 \times 10} = 0.4 \end{array} \right.$$

$$\boxed{\omega_n=10, \zeta=0.4}$$

Step 3 Resonant peak is given by

$$M_r = \frac{1}{2\zeta\sqrt{1-\zeta^2}} = 1.36$$

-02M-

Resonant frequency is given by

$$\omega_r = \omega_n \sqrt{1-2\zeta^2}$$

$$\omega_r = 8.24 \text{ rad/s} \quad -02M-$$

$$\boxed{\begin{array}{l} M_r = 1.36 \\ \omega_r = 8.24 \text{ rad/s} \end{array}}$$

Q8b. State any three rules of plotting root locus. -06M-

- ① Root locus is symmetrical about the real axis -02M-
- ② Segments of the real axis having an odd number of real axis open loop poles + zeros to their right are part of root locus. -02M-
- ③ The $(n-m)$ root locus branches that tend to ∞ , do so along straight line asymptotes making angle with real axis given by -02M-

$$\phi_A = \frac{180^\circ \cdot (2q+1)}{n-m}; \quad q = 0, 1, 2, \dots, (n-m-1)$$

Q8c

Sketch the Bode plot for open loop transfer function $G(s)H(s) = \frac{K}{s(1+0.2s)(1+0.05s)}$

Determine the value of K for gain margin (GM) of 6dB.

-10M-

Step 1 $\hat{=}$ Given open loop T.F is in Time constant form

$$G(s)H(s) = \frac{K}{s(1+0.2s)(1+0.05s)}$$

Step 2: Identifying different factors

① $\frac{1}{s}$, pole at origin, st line of slope -20dB/decade
Passing through intersection point of $\omega=1$ & 0dB

② $\frac{1}{1+0.2s}$, simple pole, $T_1 = 0.2$, $\omega_{c1} = \frac{1}{T_1} = 5$
st line of slope -20dB/decade for $\omega > 5$

③ $\frac{1}{1+0.05s}$, simple pole, $T_2 = 0.05$, $\omega_{c2} = 20$
st line of slope -20dB/decade for $\omega > 20$

-03M-

Range of ω	Resultant slope in dB/dec
$0 < \omega < 5$	-20
$5 \leq \omega < 20$	$-20 - 20 = -40$
$20 \leq \omega < \infty$	$-40 - 20 = -60$

-01M-

Step 3 Phase angle table

$$G(j\omega)H(j\omega) = \frac{K}{j\omega(1+0.2j\omega)(1+0.05j\omega)}$$

ω	$\frac{1}{j\omega}$	$-\tan^{-1} 0.2\omega$	$-\tan^{-1} 0.05\omega$	ϕ_R
0.2	-90°	-2.29	-0.57°	-92.86°
0.3	-90°	-3.43°	-0.859°	-94.29
0.5	-90°	-5.71°	-1.432°	-97.142°
2	-90°	-21.8°	-5.71°	-117.51°
5	-90°	-45°	-14.0°	-149°
7	-90°	-54.46	-19.29°	-163.7°
10	-90°	-63.43	-26.56	-180°
20	-90°	-75.9°	-45°	-211°
∞	-90°	-90°	-90°	-270°

-02M-

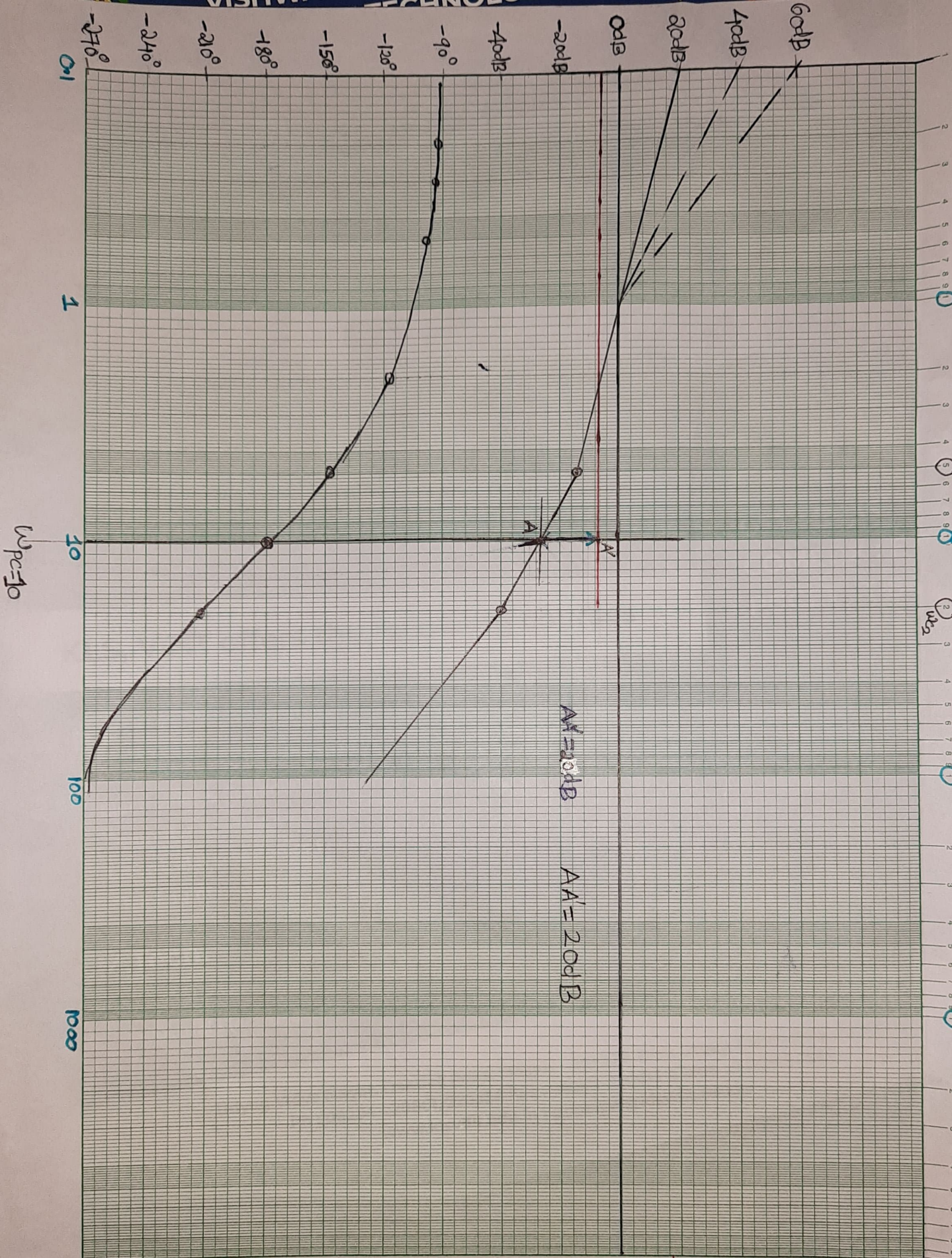
* From Bode plot graph, we observe that amount of shift required is $AA' = 20dB$

$$\therefore 20 \log_{10} K = 20 \quad ; \quad \log_{10} K = 1$$

$$K = 10^1 \quad ; \quad \boxed{K=10} \quad -01M-$$

Scale: 1 unit = 20dB
 Y-axis: 1 unit = 30°

SEMI-LOG PAPER (9 CYCLES X 1/10⁹)



Q8.C.
 (30)

-04M-

SSP

Q9a

- Module - 5 -

State Mapping Theorem and explain Nyquist Stability Criterion. (3)

-04M-

Mapping Theorem

Let $F(s)$ be a ratio of two polynomials in s . Let P be the number of poles and Z be the number of zeros of $F(s)$ that lies inside some closed contour in the s -plane, with multiplicity of poles and zeros accounted for. Let this contour be such that it does not pass through any poles or zeros of $F(s)$. This closed contour in the s -plane is then mapped into $F(s)$ plane as closed curve. The total number N of clockwise encirclements of the origin of the $F(s)$ plane, as a representative point s traces out the entire contour in the clockwise direction, is equal to $Z - P$, $N = Z - P$

-02M-

Nyquist stability criterion

If the contour Γ_{GH} of the open loop transfer function $G(s)H(s)$ corresponding to the Nyquist contour in the s -plane encircles the point $(-1 + j0)$ in the counter clockwise direction as many time as the number of righthalf s -plane poles of $G(s)H(s)$, the closed loop system is stable.

* The Mapping of the Nyquist contour into contour Γ_{GH} is carried out as follow

- ① The Mapping of the Imaginary axis is carried out by subⁿ of $s = j\omega$ in $G(s)H(s)$ [$G(j\omega)H(j\omega)$]
 - ② In physical systems ($m \leq n$), $\lim_{s=R \rightarrow \infty} G(s)H(s) = \text{real constant}$
- Thus infinite arc of Nyquist contour maps into a point on the real axis

-02M-

Q9b. Draw polar plot for $G(s)H(s) = \frac{10}{s(s+1)(s+2)}$ Find GM and PM, also comment on stability. -10M-

Step 1 Given open loop transfer function

$$G(s)H(s) = \frac{10}{s(s+1)(s+2)}$$

$$\therefore G(j\omega)H(j\omega) = \frac{10}{j\omega(j\omega+1)(j\omega+2)}$$

$$M = \frac{10}{\omega\sqrt{1+\omega^2}\sqrt{4+\omega^2}}, \quad \phi = -90^\circ - \tan^{-1}\omega - \tan^{-1}\left(\frac{\omega}{2}\right)$$

Step 2 For polar plot ω varies from 0 to ∞

$\omega = 0$	$M = \infty$	$\phi = -90^\circ$	} $-270 - (-90) = -180^\circ$ clockwise
$\omega = \infty$	$M = 0$	$\phi = -270^\circ$	

-02M-

Step 3 Let us rationalize $G(j\omega)H(j\omega)$.

$$G(j\omega)H(j\omega) = \frac{(-j\omega)(1-j\omega)(2-j\omega) \times 10}{j\omega(j\omega+1)(j\omega+2)(-j\omega)(-j\omega+1)(-j\omega+2)}$$

$$= \frac{(-j\omega)(-j\omega+1)(-j\omega+2) \times 10}{\omega^2(1+\omega^2)(\omega^2+4)}$$

$$= \frac{10(-j\omega)(-\omega^2 - 3j\omega + 2)}{\omega^2(-\omega^2+1)(\omega^2+4)} = \frac{-30\omega^2 - 10j\omega(\omega^2+2)}{\omega^2(\omega^2+1)(\omega^2+4)}$$

$$G(j\omega)H(j\omega) = \frac{-30\omega^2}{\omega^2(\omega^2+1)(\omega^2+4)} - \frac{j\omega(20-10\omega^2)}{\omega^2(\omega^2+1)(\omega^2+4)} \quad -02M-$$

Step 4 at Intersection on real axis Imag part is zero
Equating Imag to 0, we get corresponding ω , with this ω in real part we get point of intersection

$$\frac{-j\omega(20 - 10\omega^2)}{\omega^2(\omega^2 + 1)(\omega^2 + 4)} = 0 \quad | \quad \begin{aligned} 10\omega^2 &= 20 \\ \omega^2 &= 2 \\ \omega &= \sqrt{2} = 1.414 \text{ rad/s} \end{aligned}$$

$$Q|_{\omega=\omega_{pc}} = \frac{-30}{(2+1)(2+4)} = \frac{-30}{3 \times 6} = -1.66$$

Q = -1.66 -0.2M-

Steps Magnitude and Phase table

ω	M	ϕ	$G_R(j\omega)$	$G_I(j\omega)$
0.5	8.67	-130.6°	-5.6	-6.5
0.6	6.8	-137.6°	-5.02	-4.58
0.7	5.5	-144.2°	-4.4	-3.2
0.8	4.5	-150°	-3.89	-2.25
1	3.16	-161.5°	-2.996	-1.002
2	0.707	-198.43°	-0.749	0.249

-0.1M-

steps $G_M = 20 \log \frac{1}{10|Q|} = 20 \log \frac{1}{1.66} = -4.40 \text{ dB}$

PM from graph $\Rightarrow \phi_{gc} = -192^\circ$

PM = $180 + \phi_{gc} = -12^\circ$

Both G_M and PM are negative system is
Unstable

-0.1M-

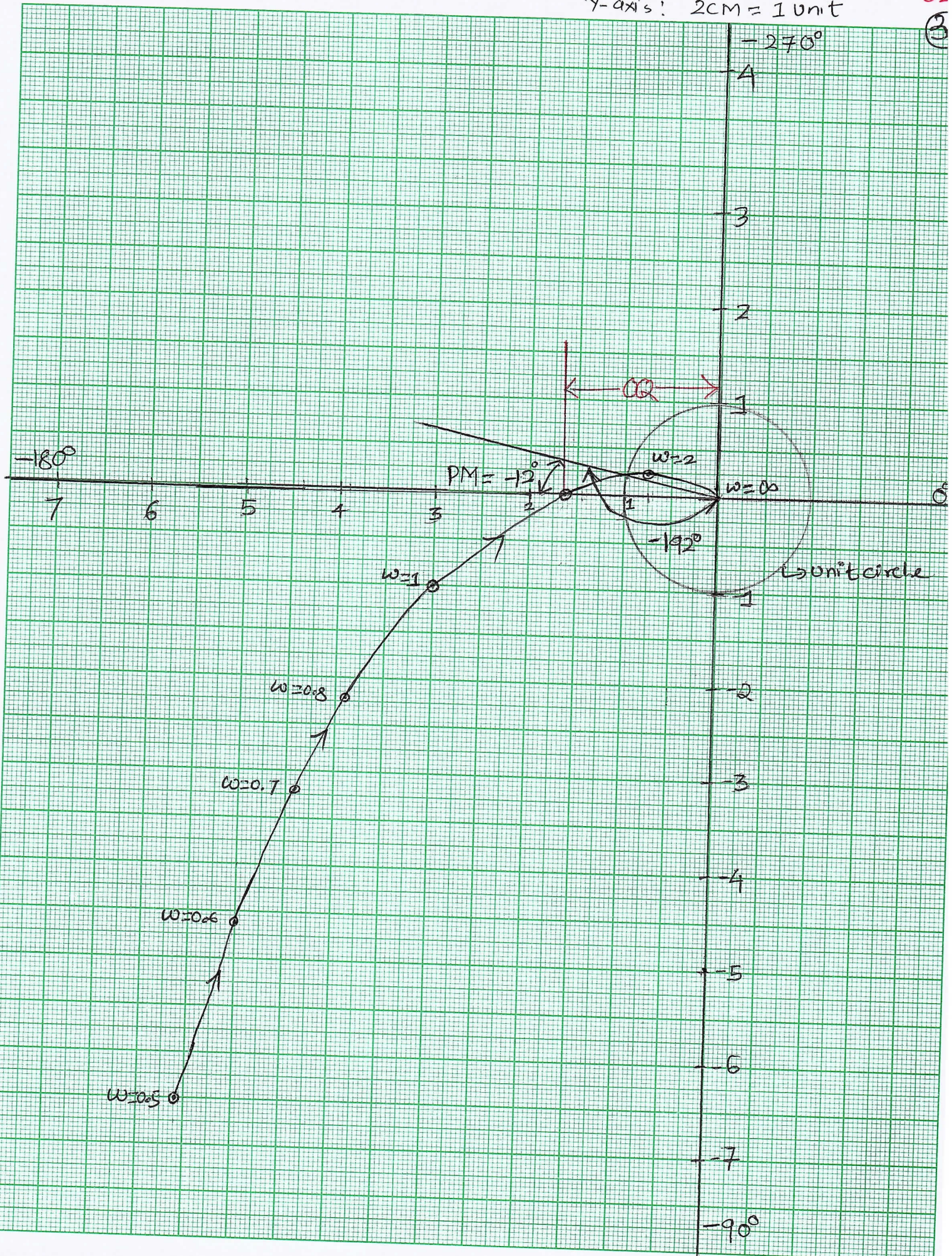
Q9.b

Scale: X-axis : 2CM = 1 unit

Y-axis : 2CM = 1 unit

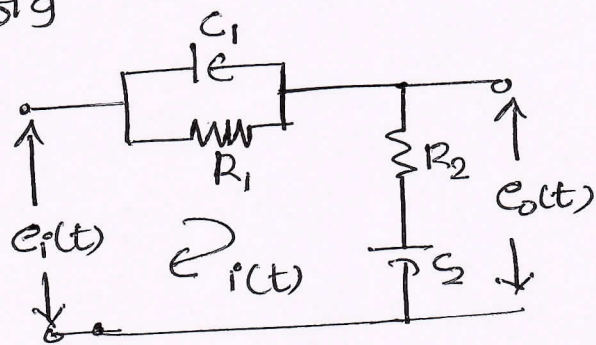
-02

3



Q9C. Explain Lead Lag compensating network -06M-

A Lag-Lead compensator is a combination of lag and lead compensators. An electric network which acts as lag-lead compensator is shown in below fig



-02M-

* The Transfer function of above network is

$$\frac{E_o(s)}{E_i(s)} = \frac{(1+T_1s)(1+T_2s)}{\left(1+\frac{T_1s}{\beta}\right)(1+T_2s)}$$

Where $T_1 = R_1C_1$, $T_2 = R_2C_2$, $\frac{\beta}{T_1} + \frac{1}{\beta T_2} = \frac{1}{R_1C_1} + \frac{1}{R_2C_2}$, $\alpha \beta = 1$

* The phase lead portion involving T_1 , adds phase lead angle while the phase lag portion involving T_2 provide attenuation near and above the gain crossover frequency -02M-

* Effects of Lag-Lead compensator

- * Lag-lead compensator is used when both fast response and good static accuracy are desired.
- * Use of Lag-lead compensator increases the low frequency gain which improves the steady state
- * It increases bandwidth of the system, making the system response very fast.

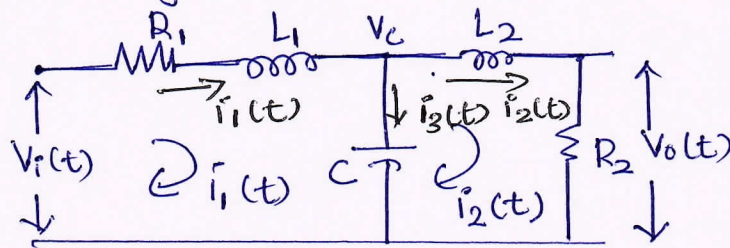
-02M-

-OR-

Q10a What are the advantages of state-space analysis

- ① State-space analysis includes the effect of all initial conditions -04M-
- ② It is useful to determine the time-domain response of non-linear systems effectively -01M-
- ③ state equations involving matrix algebra are highly compatible with simulation on digital computers. -01M-
- ④ MIMO systems can be easily represented and analyzed using state variables. -01M-

Q10b Obtain the state equations for the electrical Network shown in fig 10.b -08M-



Step 1 ∴ Label all branch current

Step 2 ∴ derivative relationship for storage element

$$i_3 = C \frac{dV_c(t)}{dt} \quad - (1)$$

$$V_{L_1} = L_1 \frac{di_1(t)}{dt} \quad - (2)$$

$$V_{L_2} = L_2 \frac{di_2(t)}{dt} \quad - (3)$$

* State variable are i_1, i_2, V_c -02M-

Step 3: Apply KCL at V_C

$$i_1(t) = i_3(t) + i_2(t)$$

$$\boxed{i_3 = i_1 - i_2} \quad \text{--- (4) ---}$$

KVL to loop 1

$$-V_i + R i_1(t) + V_{L_1} + V_C = 0$$

$$\boxed{V_{L_1} = V_i - V_C - R i_1} \quad \text{--- (5) ---}$$

KVL to loop 2 $-V_C + V_{L_2} + V_o = 0$

$$\boxed{V_{L_2} = V_C - i_2 R_2} \quad \text{--- (6) ---}$$

--- 03M ---

Subⁿ of 4, 5, 6 in 1, 2, 3

$$V_C^o = \frac{1}{C} i_1 - \frac{1}{C} i_2 \quad | \quad i_2^o = \frac{V_C}{L_2} - i_2 \frac{R_2}{L_2}$$

$$i_1^o = \frac{V_i}{L_1} - \frac{V_C}{L_1} - \frac{R i_1}{L_1} \quad | \quad$$

--- 02M ---

Step 4 output equation $V_o = i_2 R_2$

state eqⁿ and output eqⁿ in vector-matrix form

$$\begin{bmatrix} i_1^o \\ i_2^o \\ V_C \end{bmatrix} = \begin{bmatrix} -\frac{R}{L_1} & 0 & -\frac{1}{L_1} \\ 0 & -\frac{R_2}{L_2} & 0 \\ \frac{1}{C} & -\frac{1}{C} & 0 \end{bmatrix} \begin{bmatrix} i_1 \\ i_2 \\ V_C \end{bmatrix} + \begin{bmatrix} \frac{V_i}{L_1} \\ 0 \\ 0 \end{bmatrix} V_i^o$$

$$V_o = \begin{bmatrix} 0 & R_2 & 0 \end{bmatrix} \begin{bmatrix} i_1 \\ i_2 \\ V_C \end{bmatrix} \quad \text{--- 01M ---}$$

Q10: Compute the STM for the given system matrix

$A = \begin{bmatrix} 1 & 0 \\ 1 & 1 \end{bmatrix}$ using Laplace approach technique. -08M-

Step 1 $[sI - A] = \begin{bmatrix} s-1 & 0 \\ -1 & s-1 \end{bmatrix}$ -01M-

Step 2 $[sI - A]^{-1} = \frac{\text{Adj}[sI - A]}{|sI - A|}$

$\text{Adj}[sI - A] = \begin{bmatrix} s-1 & 1 \\ 0 & s-1 \end{bmatrix}^T = \begin{bmatrix} s-1 & 0 \\ 1 & s-1 \end{bmatrix}$

$|sI - A| = (s-1)^2$

$\therefore [sI - A]^{-1} = \frac{\begin{bmatrix} s-1 & 0 \\ 1 & s-1 \end{bmatrix}}{(s-1)^2} = \begin{bmatrix} \frac{1}{s-1} & 0 \\ \frac{1}{(s-1)^2} & \frac{1}{s-1} \end{bmatrix}$ -04M-

Step 3 State transition Matrix STM

$e^{At} = \mathcal{L}^{-1} [sI - A]^{-1} = \mathcal{L}^{-1} \begin{bmatrix} \frac{1}{s-1} & 0 \\ \frac{1}{(s-1)^2} & \frac{1}{s-1} \end{bmatrix}$

$e^{At} = \begin{bmatrix} e^t & 0 \\ t e^t & e^t \end{bmatrix}$ -03M-