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## Fourth Semester B.E. Degree Examination, July/August 2022 Analysis of Determinate Structures

Max. Marks: 100

Time: 3 hrs.

*Note: Answer any FIVE full questions, choosing ONE full question from each module.*

### Module-1

- 1 a. Explain with examples statically determinate and indeterminate structures.  
 b. Find the Static and Kinematic indeterminacies of the following structures.

5  
(08 Marks)

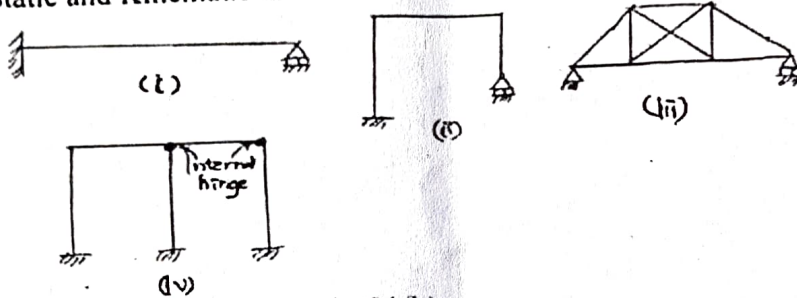


Fig.Q1(b)

12  
(12 Marks)

OR

- 2 a. What do you mean by influence line diagram and state its applications.  
 b. Draw ILD for  
 (i) Reactions at supports of a simply supported beam.  
 (ii) Shear force of a simply supported beam carrying concentrated unit load.

(08 Marks)

(12 Marks)

### Module-2

- 3 a. Two point loads 4 kN and 6 kN spaced 6m apart cross a girder of 16m span, the 4 kN load leading from left to right. Construct the maximum SF and BM diagrams stating the absolute maximum values. [Fig.Q3(a)].

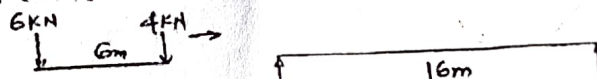


Fig.Q3(a)

(10 Marks)

- b. Draw the influence line for SF and BM at a section 5m from the left hand support of a simply supported beam 25m span. Hence calculate maximum shear force and BM at this section due to uniformly distributed load of 1 kN/m, 8m long. [Refer Fig.Q3(b)]

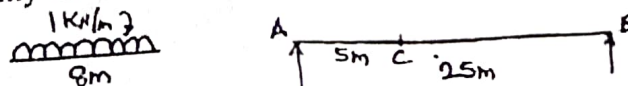


Fig.Q3(b)

(10 Marks)

OR

- 4 A simply supported beam of span 20m is subjected to a set of loads of magnitude of 20 kN, 30 kN, 15 kN and 10 kN spaced as shown with 10 kN leading. Determine the maximum BM at a section 5m from the left end and also the absolute maximum BM developed in the beam. [Refer Fig.Q4]

20 (20 Marks)

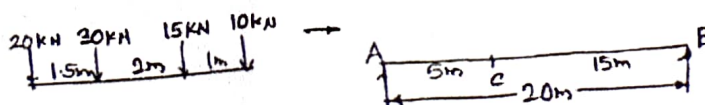


Fig.Q4

**Module-3**

- 5 a. Determine the slope and deflection at the free end of a cantilever beam loaded as shown in the Fig.Q5(a). Take  $EI = 4 \times 10^5 \text{ kNm}^2$ . Use moment area method.

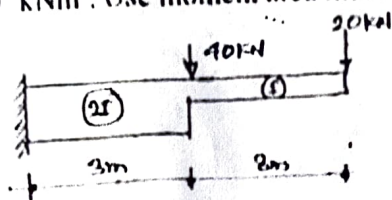


Fig.Q5(a)

10  
(10 Marks)

- b. Determine the slope at C and deflection at D of a simply supported beam shown in Fig.Q5(b). Take  $E = 200 \text{ GPa}$ ,  $I = 2 \times 10^6 \text{ mm}^4$ . Use conjugate beam method.

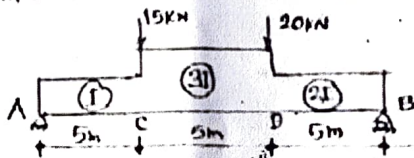


Fig.Q5(b)

10  
(10 Marks)

OR

- 6 a. Determine the slope at the supports and deflection at the centre of a simply supported beam with a point load  $W$  at its mid span. Use moment area method. (10 Marks)  
 b. Determine the slope at the supports and deflection at the centre of a simply supported beam with uniformly distributed load of  $W/m$  over the entire span. Use moment area method. (10 Marks)

**Module-4**

- 7 a. Derive the expression for strain energy stored in an prismatic element subjected to pure bending moment. (08 Marks)  
 b. Determine the vertical deflection at C of a bent frame shown in the Fig.Q7(b). Use Castigliano's approach. Take  $E = 200 \text{ GPa}$ ,  $I = 80 \times 10^7 \text{ mm}^4$ .

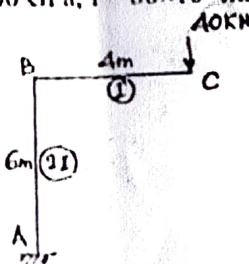


Fig.Q7(b)

6  
12

(12 Marks)

OR

- 8 Determine the vertical and horizontal deflection of the point C, of the pin jointed frame shown in Fig.Q8. The cross sectional area of AB = 100 sqmm and BC and AC are 150 sqmm. Take  $E = 2 \times 10^5 \text{ N/mm}^2$ .

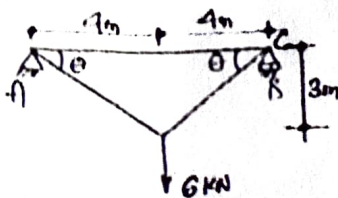


Fig.Q8



(20 Marks)

Module-5

- 9 A three hinged parabolic arch of 20m span with 4m central rise carries a point load of 4kN at 4m horizontally from the left hinge. Calculate the normal thrust and radial shear at a section just after the load. Also calculate the maximum positive and negative BM. Sketch BMD. (20 Marks)

OR

- 10 A cable is of uniform section is suspended between two supports 100m apart. It carries a uniformly distributed load of 10 kN/m spread over the horizontal span. Find
- Maximum and minimum tension in the cable.
  - Minimum cross sectional area of the cable required if the allowable stress is 300 MPa.
  - Length of the cable.
- (20 Marks)

\* \* \* \* \*



Sub: Analysis of Determinate Structures (18CV42)

July/August - 2022.

Q1 @ Statically Determinate Structure:

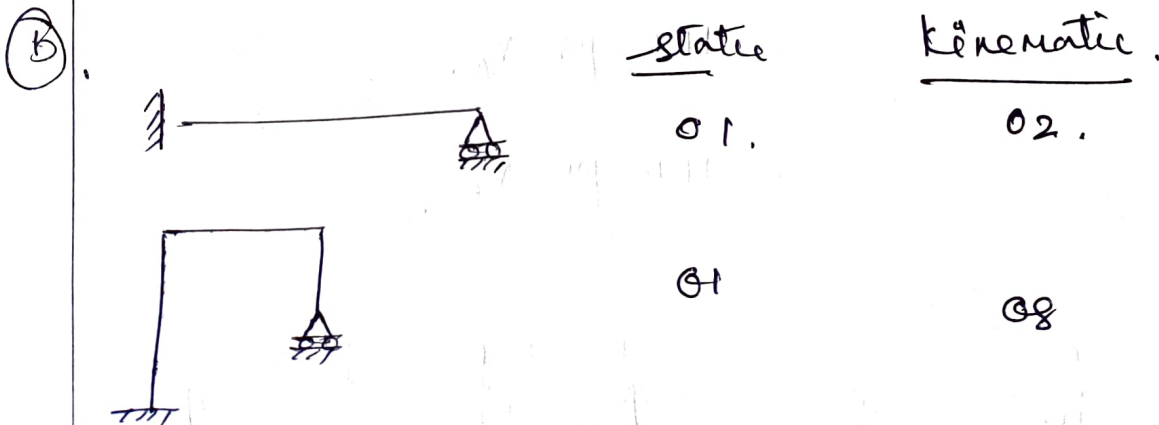
A structure said to be statically determinate when the forces & bending moments are determined by static equilibrium equations. Usually in this case if support is removed, the structure collapses.

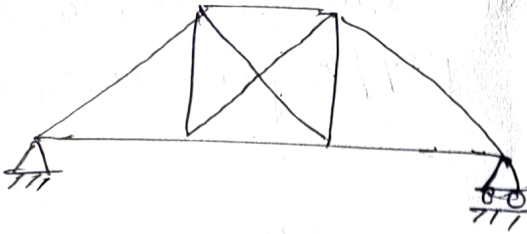
- Ex: ① Cantilever beam.  
 ② Simply supported beam.  
 ③ Three hinged arch.

⑥ Indeterminate Structures:

These structures contain same extra reaction forces and equilibrium conditions are not sufficient to determine forces. Thus equilibrium equations are to be supplemented by equation of compatibility of displacement.

- Ex: ① propped cantilever beam  
 ② fixed beam  
 ③ portal frames.



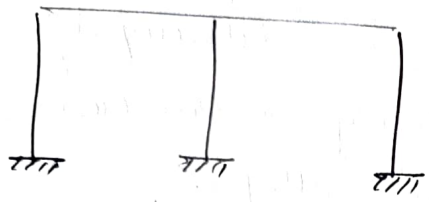


Static

Kinematic

01.

09.



06.

09.

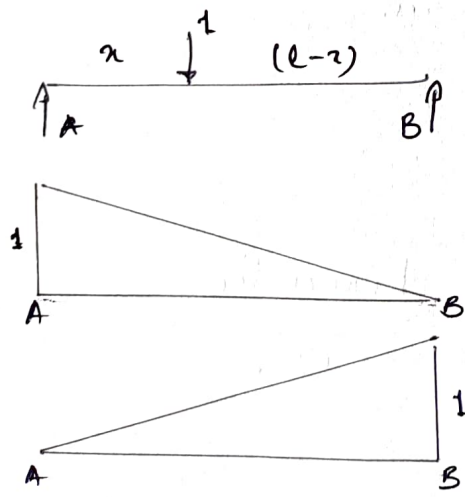
2Q.

Influence line is a curve showing the variation of any resultant action, shear, bending moment, deflection or slope at a particular point in a structure under the influence of a unit load that rolls across span.

Its applications:

- (i) Can be used all types of loads.
- (ii) For designing can use Max value of SF & BM.
- (iii) Suitable for both stationary & moving loads.
- (iv) Shear stress reversal in panels can be assessed in truss.

(b)



Reactions:

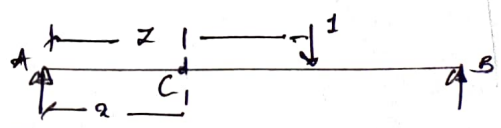
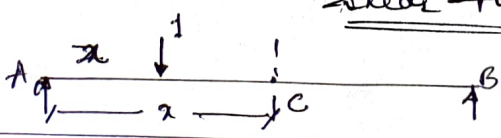
$$\sum V = 0 \quad V_A + V_B = 1.$$

$$\sum M_A = 0 \quad V_B = x/L \quad V_A = 1 - x/L$$

← ILD for  $V_A$ .

← ILD for  $V_B$ .

Shear force



$$V_B = -2/L$$

$$z=0$$

$$SF=0$$

$$\textcircled{1} 0 < z < L$$

$$z=L$$

$$SF = -2/L$$

$\textcircled{ii}$

$$L < z < L$$

$$V_A = 1 - \frac{(L-z)}{L}$$

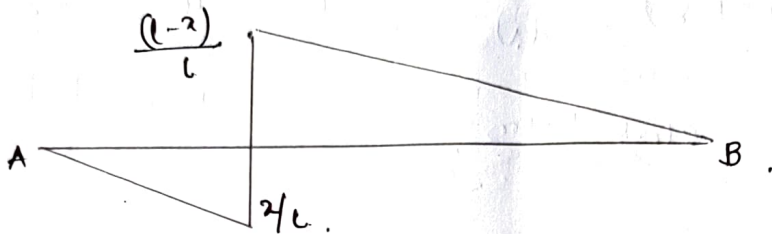
$$z=L$$

$$SF = (L-z)/L$$

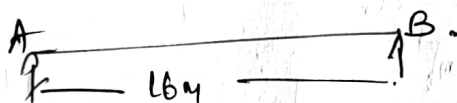
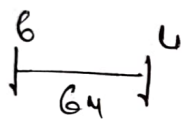
$$z=L$$

$$SF = 0$$

$$\frac{(L-z)}{L}$$

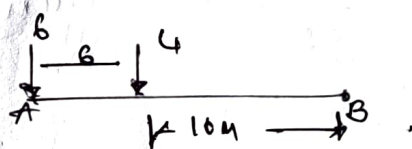


3a



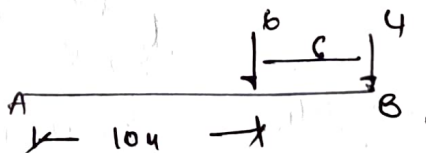
the shear force diagram

$$R_B = \frac{6 \times 6}{16} = 1.5$$



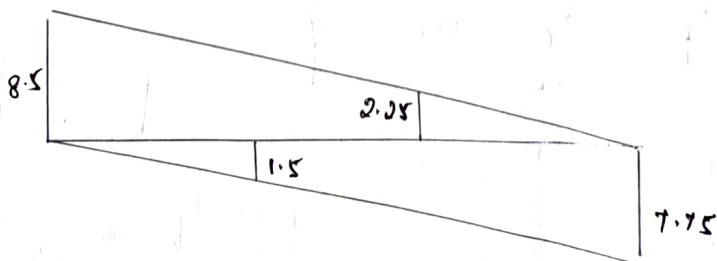
$$R_B = \frac{6 \times 10 + 4 \times 6}{16} = 7.75$$

Max the shear



$$R_A = \frac{6 \times 6}{16} = 2.25$$

$$R_A = \frac{4 \times 10 + 6 \times 6}{16} = 8.5$$



A M N S

for BM

$$M_A = \frac{6 \times 6}{6+4} = 3.64$$

$$M_B = \frac{4 \times 6}{6+4} = 2.44$$

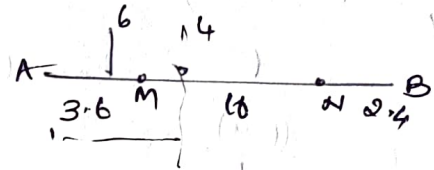
$$MN = 16 - 3.6 - 2.4 = 10 \text{ m}$$

① case 1  $y_2 (AB - AM) = 2 = 9.8 \text{ m}$

$$R_B = (4 \times 9.8 + 6 \times 3.8) / 16 = 3.875 \text{ kN/m}, \quad M = 24 \text{ kNm}$$

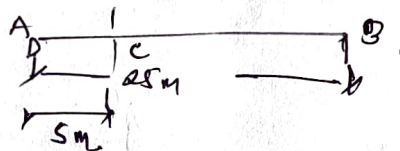
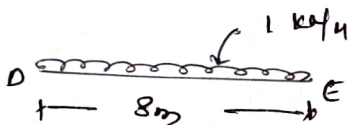
②  $x_1 = 6.8 \text{ m from A}$

$$R_B = \frac{(6 \times 0.6 + 4 \times 6.8)}{16} = 2 \text{ kN}$$

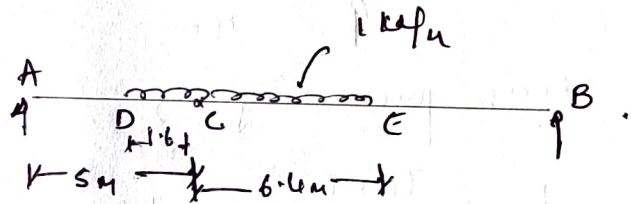


$$M_{max} = 2 \times 12.4 = 24.8 \text{ kNm}$$

3 b



$$CF = \frac{8 \times 20}{25} = 6.4 \text{ m}$$



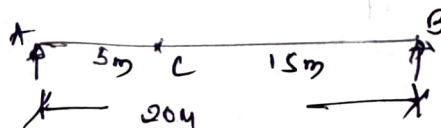
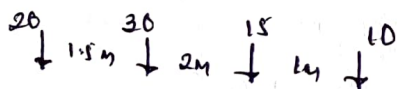
where  $x = 9$   $V_{max} = \frac{-w x^2}{2L} = -\frac{1 \times 8^2}{2 \times 25} = 1.28 \text{ kN}$

$x = L$   $V_{max} = \frac{-w a}{L} (L - a/2) = -\frac{1 \times 8}{25} (25 - 8/2) = 6.12$

$$R_B = \frac{1 \times 8 \times 7.4}{25} = 2.37 \text{ kN}$$

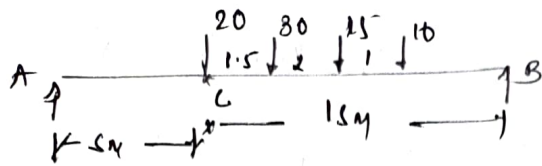
$$M_{max} = 2.37 \times 20 - 1 \times 6.4^2 / 2 = 26.92 \text{ kNm}$$

④



Load	(P <sub>1</sub> )	(P <sub>2</sub> )	Remark
10	6.5/5	10/15	
30	50/5 = 10	25/15 → 1.67	
20	20/5 = 4	55/15 → 3.67	





$$V_A = (10 \times 10.5 + 15 \times 11.5 + 30 \times 13.5 + 20 \times 15) / 20$$

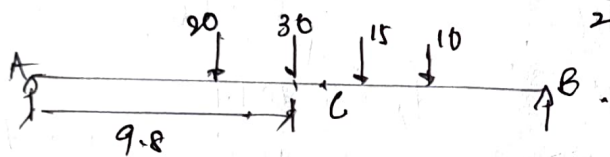
$$\therefore V_A = 49.125 \text{ kN}$$

$$M_C = 265.62 \text{ kNm}$$

Abscissa BM

$$\bar{x} \text{ from } 20 = \frac{30 \times 1.5 + 15 \times 3.5 + 10 \times 4.5}{20 + 30 + 15 + 10} = 1.9$$

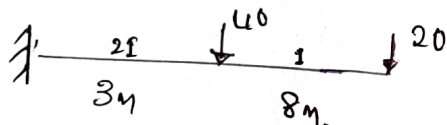
Distance of 30kN from 'A' =  $\frac{20}{2} - \frac{1}{2}(1.9 - 1.5) = 9.8 \text{ m}$



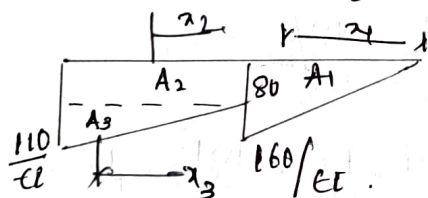
$$V_A = \frac{10 \times 7.2 + 15 \times 8.2 + 30 \times 10.2 + 20 \times 11.7}{20} = 33.51 \text{ kN}$$

$$M_C = 33.51 \times 10 - 20 \times 1.7 - 30 \times 0.2 = 295.1 \text{ kNm}$$

5 @



$$EI = 4 \times 10^5 \text{ kNm}^2$$



$$A_1 = 640, \quad x_1 = 5.33 = 3413$$

$$A_2 = 240, \quad x_2 = 9.5 = 2280$$

$$A_3 = 45, \quad x_3 = 10 = 450$$

$$A_4 = 90, \quad x_4 = 10 = 900$$

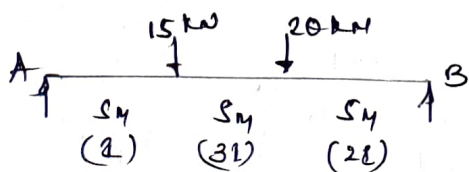
$$\underline{\underline{1015 / EI}}$$

$$\underline{\underline{7043 / EI}}$$

$$Q = \frac{1015}{4 \times 10^5} = 2.53$$

$$\Delta = \frac{7043}{4 \times 10^5} = 1.76$$

6



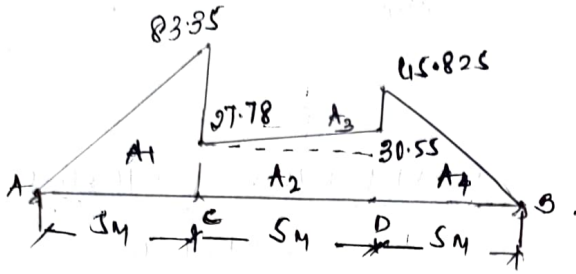
$$E = 200 \text{ GPa}$$

$$I = 2 \times 10^6 \text{ mm}^4$$



$$R_B = \frac{12 \times 5 + 20 \times 10}{15} = 18.33, \quad R_A = 11.67$$

$$M_C = 11.67 \times 5 = 58.35 \quad \& \quad M_D = 18.33 \times 5 = 91.65$$



$$A_1 = 208.37 \times 11.67 = 2431.7$$

$$A_2 = 138.90 \times 7.5 = 1041.75$$

$$A_3 = 69.25 \times 6.67 = 461.9$$

$$A_u = \frac{114.56 \times 3.33}{488.76} = \frac{381.48}{3901.12}$$

$$R_A = \frac{3901.12}{15} = 260.07$$

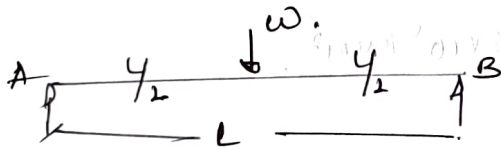
$$R_B = 208.76$$

$$Q_C = 260.07 - 208.37 = 51.63$$

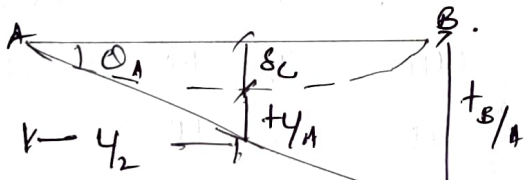
$$\Delta_D = \frac{208.76 \times 5 - 381.48}{EI} = \frac{662.32}{EI}$$

$$\Delta_D = \frac{662.32 \times 10^9}{200 \times 2 \times 10^6} = 1.655$$

60



$$R_A = R_B = wL/2$$



$$\theta_{B/A} = \frac{wL}{2} \times \frac{L}{3} = \frac{wL^2}{6EI} \times \frac{L}{2}$$

$$= \frac{1}{2} \times \frac{L}{2} \times w \times \frac{L}{2} = \frac{wL^2}{4}$$

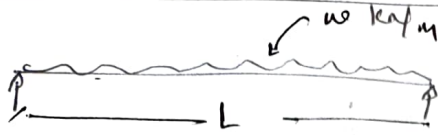
$$\theta_{B/A} = \frac{wL^3}{12} - \frac{wL^3}{18} = \frac{wL^3}{18EI}$$

$$Q_A = \frac{wL^3/18}{L} = \frac{wL^2}{18EI}$$

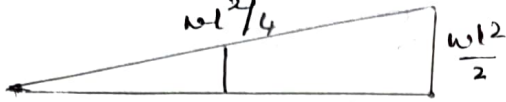
$$\theta_{C/A} = \frac{1}{2} \times \frac{L}{2} \times \frac{wL}{4} = \frac{wL^2}{16EI} \times \frac{L}{2} = \frac{wL^3}{32EI}$$

$$\delta_c = \left( \frac{wL^2}{16} \times \frac{L}{2} - \frac{wL^3}{32EI} \right) = \frac{2wL^3}{32EI} = \frac{wL^3}{96EI}$$

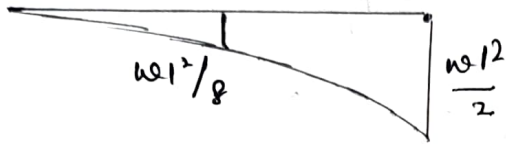
Q6



$$R_A = R_B = wL/2$$



$$f_{s/A} = \frac{1}{2} \times L \times \frac{wL^2}{2} = \frac{1}{3} \times L \times wL^2/2$$



$$= \left(\frac{wL^3}{4}\right) \frac{1}{3} - \frac{wL^3}{6} = \left(\frac{1}{4}\right)$$

$$f_{s/A} = \frac{wL^4}{24EL}$$

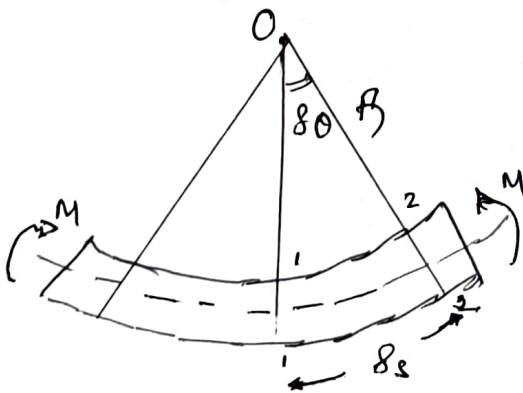
$$Q_A = \frac{wL^6}{24EL} = \frac{wL^3}{24EL}$$

$$f_{y/A} = \left(\frac{1}{2} \times \frac{1}{2} \times \frac{wL^2}{4}\right) \cdot \frac{1}{3 \times 2} - \left(\frac{1}{3} \times \frac{1}{2} \times \frac{wL^2}{8}\right) \cdot \left(\frac{1}{6} \times \frac{1}{2}\right)$$

$$= \frac{wL^6}{96EL} - \frac{wL^6}{384EL} = \frac{3wL^6}{384EL}$$

$$\delta_c = Q_A \cdot \frac{1}{2} - f_{y/A} \Rightarrow \frac{wL^3}{24EL} \times \frac{1}{2} - \frac{3wL^6}{384EL} = \frac{5wL^6}{384EL}$$

Module - 4



$$\delta_{cb} = \frac{1}{2} M \cdot \delta_c$$

from fig  $R\delta_c = \delta_c$

$$\delta_{cb} = \frac{1}{2} M \cdot \frac{\delta_c}{R}$$

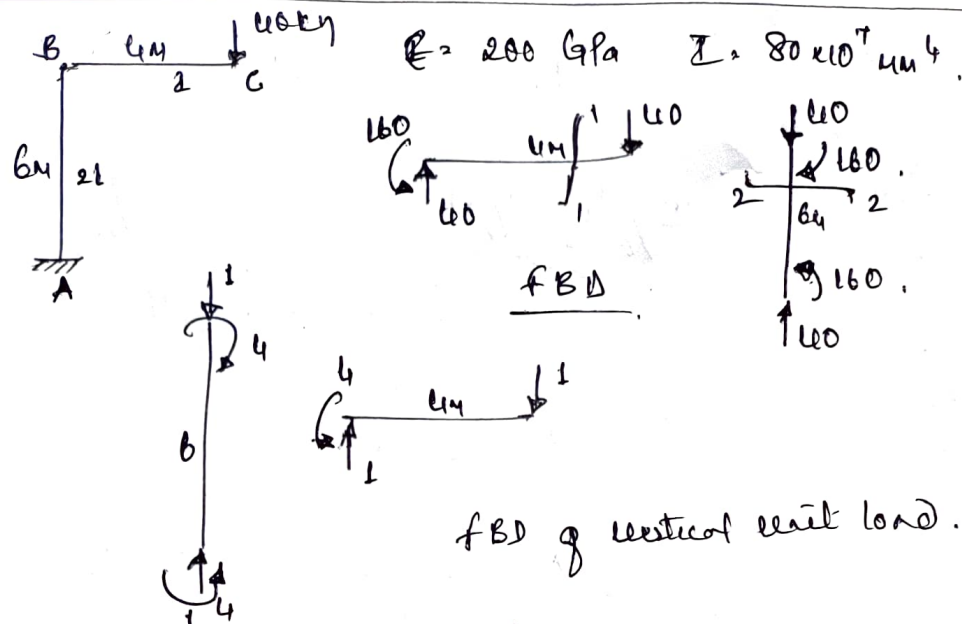
from theory of bending

$$\frac{M}{I} = \frac{f}{R} = \frac{f}{y} \Rightarrow \frac{1}{R} = \frac{M}{EI}$$

$$\delta_{cb} = \frac{1}{2} M \cdot \frac{M}{EI} \cdot \delta_c \Rightarrow \delta_{cb} = \int \frac{M^2}{2EI} \delta_c$$

7a

7(b)

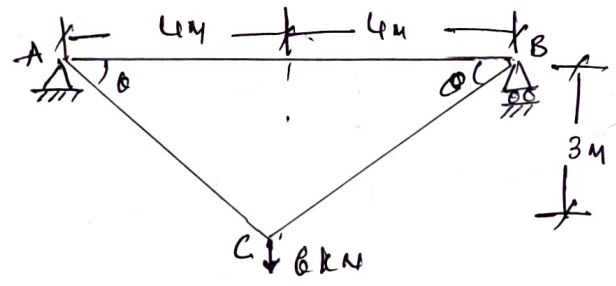


Member	origin	limit	M	m	l
BC	C	0-4	-40x	-x	4
AB	B	0-6	-160	-4	6

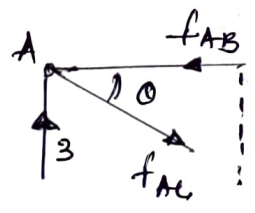
$$\Delta = \int \frac{M \cdot m}{EI} dx = \int_0^4 \frac{(-40x) \cdot (-x)}{EI} dx + \int_0^6 \frac{-160 \cdot (-4)}{2EI} dx$$

$$\Delta = \frac{40 \cdot 2^3}{EI \cdot 3} \Big|_0^4 + 320 \left[ \frac{x}{2} \right]_0^6 \Rightarrow \frac{2773.3}{EI} = 1.733 \text{ mm}$$

8



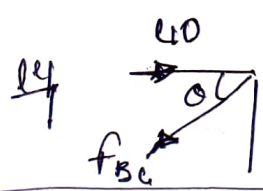
$$V_A = V_B = \frac{6}{2} = 3 \text{ kN}$$



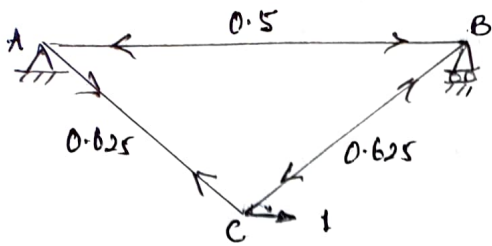
$$\tan \theta = \frac{3}{4} \therefore \theta = 36.86^\circ \quad \sin \theta = 0.6 \quad \cos \theta = 0.8$$

$$\sum V = 0 \quad f_{AC} \sin \theta = 3 \quad \therefore f_{AC} = 5 \text{ kN}$$

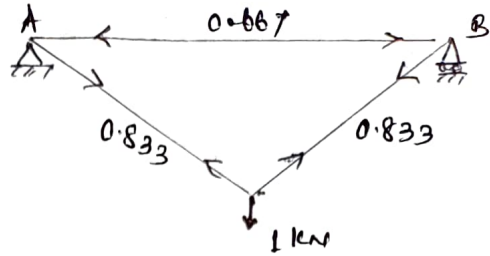
$$\sum H = 0 \quad f_{AB} = 5 \cos \theta = 4 \text{ kN}$$



$$\therefore f_{BC} = \frac{40}{\cos \theta} = 5 \text{ kN}$$



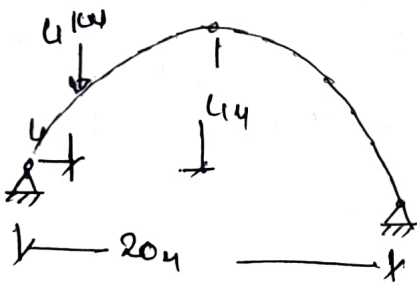
Applying horiz load 1 kN



Applying vertical load 1 kN.

members	length	Area	$\phi$ (kN)	$k$	$kL/A$	$k'$	$k'L/A$
AB	8000	100	-4	-0.667	23.04	-0.5	160
BC	5000	150	5	0.833	138.83	0.625	104.1
AC	5000	150	5	0.833	138.83	0.625	104.1
<u>491.10</u>							<u>160</u>

$$\Delta_{V_C} = \frac{491.10}{200} = 2.455 \text{ mm} \quad \Delta_{V_H} = \frac{160}{200} = 0.8 \text{ mm}$$



$$\sum V = V_A + V_B = 4 \text{ kN}$$

$$\therefore V_B = \frac{4 \times 4}{20} = 0.8 \text{ kN}$$

$$V_A = 4 - 0.8 = 3.2 \text{ kN}$$

$$\sum M_C = 0, \quad 3.2 \times 10 - 4 \times 6 - H_A \times 4 = 0 \quad \therefore H_A = 2 \text{ kN}$$

$$y = \frac{4 \times 2}{12} x(1-x) = \frac{4 \times 6}{20^2} (20-x)x = 0.8x - 0.04x^2$$

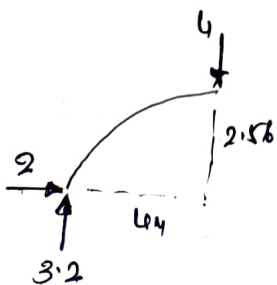
$$2y = 2.56 \text{ m}$$

$$\frac{\partial y}{\partial x} = 0.8 - 0.08x$$

$$\tan \theta = 0.48$$

$$\sin \theta = 0.432$$

$$\cos \theta = 0.901$$



$$NS = V \sin \theta + H \cos \theta$$

$$= 0.8 \times 0.432 + 2.0 \times 0.901 = 2.14$$

$$RS = V \cos \theta - H \sin \theta$$

$$= 0.8 \times 0.901 - 2.0 \times 0.432 = 0.58$$

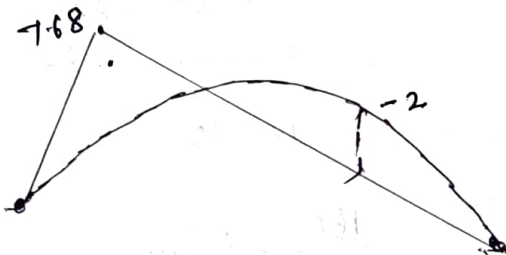


Max +ve BM  $3.2 \times 4 = 2 \times 2.56 = 7.68$

Max -ve BM  $0.8 \times (20-x) - 2(0.8x - 0.04x^2)$   
 $= 16 - 0.8x - 1.6x + 0.08x^2$

$\frac{\partial M}{\partial x} = 0 - 0.8x - 2.4 \quad \& \quad x = 15 \text{ m}$

-ve BM  $M_x = 0.8(20-15) - 2(0.8 \times 15 - 0.04(15)^2)$   
 $= -2 \text{ kNm}$



10

Assume depth as 10m.

$V_A = V_B = \frac{10 \times 100}{2} = 500 \text{ kN}$

$H = \frac{w l^2}{8h} = \frac{10 \times 100^2}{8 \times 10} = 1250 \text{ kN}$



$T_{\text{max}} = \frac{w l}{2} \sqrt{1 + \frac{l^2}{16h^2}} = \sqrt{V^2 + H^2} = 1346.3 \text{ kN}$

$T_{\text{min}} = H = 1250 \text{ kN}$

$\tan \theta = \frac{V}{H}$   
 $\theta = 21^\circ 48'$

min  $\sigma_c$  area =  $\frac{1346.3}{300} = 4.48 \times 10^3 \text{ mm}^2$

length of cable =  $100 + \frac{8}{3} \times \frac{10^2}{100} = 102.67 \text{ m} \quad \left[ l + \frac{8}{3} \frac{d^2}{L} \right]$

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 (G.V. Chalapati)

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Length